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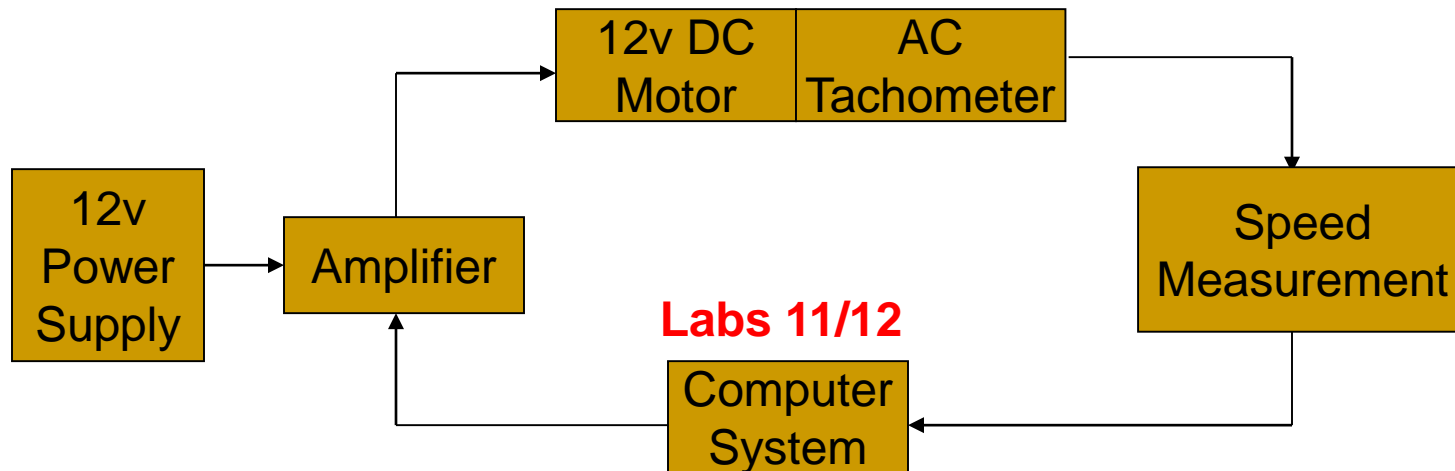
# Lab 12. Speed Control of a D.C. motor

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## Controller Design

# Motor Speed Control Project

1. Generate PWM waveform
2. Amplify the waveform to drive the motor
3. Measure motor speed
4. Measure motor parameters
5. **Control speed with a PID controller**

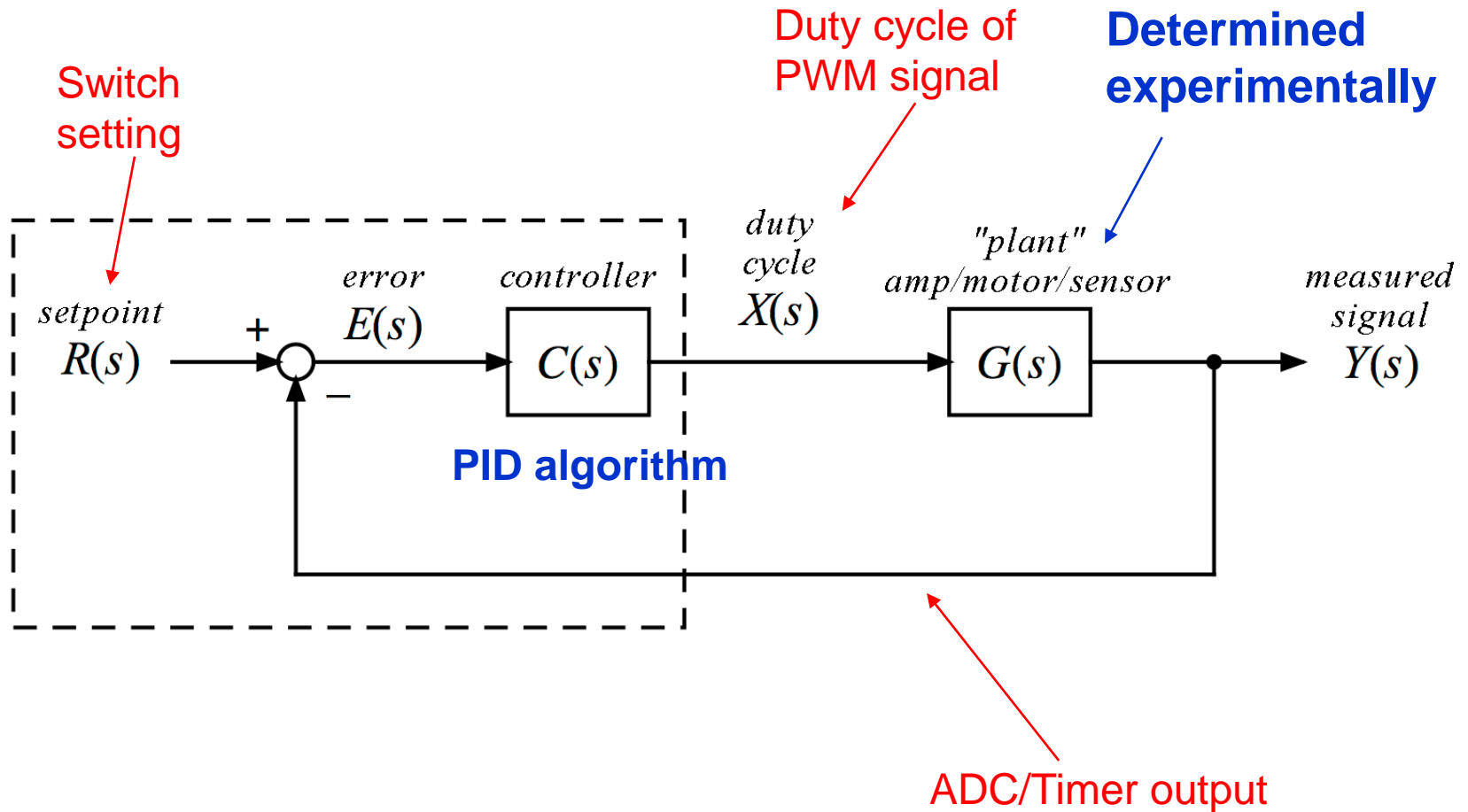


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# Technical goals for lab

- Design a PID controller to regulate the motor speed
    - Design in Simulink
    - Implement in software
  
  - Demonstrate improvement in motor performance with the PID controller
    - Faster rise time (50% or more)
    - No steady-state error
    - Minimal overshoot of target speed
    - Fast settling time when changing speeds
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# Simplified system model



# PID Controller

- Controller computes “control action”  $a(t)$ 
  - PWM signal duty cycle
- Compensate for error,  $e(t)$ , between set point and measured speed
- PID = *Proportional, Integral, Derivative*
  - Control action  $a(t)$  based on three “terms”:
    - P term - proportional to  $e(t)$
    - I term - proportional to the integral of  $e(t)$
    - D term - proportional to the derivative of  $e(t)$

# PID Controller Design

Continuous time domain:

$$a(t) = K_P e(t) + K_I \int_0^t e(t) dt + K_D \frac{de(t)}{dt}$$

$e(t)$  = error detected at time  $t$

$a(t)$  = control action computed at time  $t$

$K_P, K_I, K_D$  = constants

LaPlace transform/Controller transfer function:

$$C(s) = \frac{A(s)}{E(s)} = K_P + \frac{K_I}{s} + K_D s$$

# PID Controller Design

Discrete time domain:

$$a(nT) = K_P e(nT) + K_I \sum_{i=0}^n \frac{e(iT) + e(iT - T)}{2} + K_D \frac{e(nT) - e(nT - T)}{T}$$

$T$  = sampling time

$n$  = sample number

$e(nT)$  = error computed at  $n$ th sampling interval

$a(nT)$  = control action computed at  $n$ th sampling interval

$K_P, K_I, K_D$  = constants

$z$  transform:

$$C(z) = \frac{A(z)}{E(z)} = K_P + K_I \frac{T}{2} \left[ \frac{z+1}{z-1} \right] + K_D \frac{1}{T} \left[ \frac{z-1}{z} \right]$$

# Proportional Control

- Controller produces a control action that is proportional to the error at any given time

$$a(t) = K_p e(t)$$

- Advantages:

- Simple to implement
- Larger  $K_p$  causes greater system response to a given error  $e(t)$  (possibly improve system performance)

- Disadvantages:

- some steady-state error is required to have a control action
- can overshoot set point and possibly produce unstable behavior
- controller reacts to high-frequency “noise” in  $e(t)$



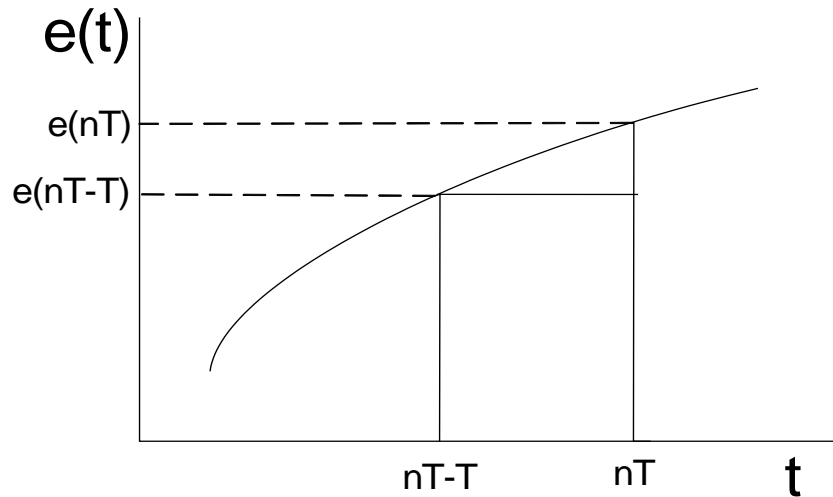
# Integral Control

- Controller produces a control action proportional to the integral of the error (area under the error curve)

$$a(t) = K_I \int_0^t e(t) dt$$

- This term determines the steady-state control action when  $e(t)=0$  (P and D terms are both 0)
- Advantages:
  - eliminates steady state error
  - dampens response to high frequency noise on  $e(t)$
- Disadvantages:
  - slows system response
  - can contribute to overshoot

# Convert integral term to discrete form



Compute the area under the curve

Consider integral up to previous and current sample times:

$$y(t) = K_I \int_0^t e(t) dt = K_I \int_0^{nT-T} e(t) dt + K_I \int_{nT-T}^{nT} e(t) dt$$

(Red arrows point from the limits  $nT-T$  and  $nT$  in the second integral to the text below)

$$y(nT) = y(nT - T) + \Delta y(nT, nT - T)$$

area up to  
 $t = nT - T$

incremental area  
from  $(nT - T)$  to  $nT$

# Convert integral term to discrete form (2)

Area under the curve between  $(nT-T)$  and  $nT$ :  
(approximate as a trapezoid)

rectangle part

triangle part

$$\begin{aligned}\Delta y(nT - T) &= K_I T e(nT - T) + K_I \left(\frac{1}{2}\right) T [e(nT) - e(nT - T)] \\ &= \frac{K_I T}{2} [e(nT) + e(nT - T)]\end{aligned}$$

Add this to  $y(nT-T)$ :

$$y(nT) = y(nT - T) + \frac{K_I T}{2} [e(nT) + e(nT - T)]$$

# Discrete transfer function of the integral term

Difference equation:

$$y(nT) = y(nT - T) + \frac{K_I T}{2} [e(nT) + e(nT - T)]$$

z transform:

$$Y(z) = Y(z)z^{-1} + \frac{K_I T}{2} [E(z) + E(z)z^{-1}]$$

Transfer function:

$$\frac{Y(z)}{E(z)} = K_I \frac{T}{2} \left[ \frac{1 + z^{-1}}{1 - z^{-1}} \right] = K_I \frac{T}{2} \left[ \frac{z + 1}{z - 1} \right]$$

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# Derivative control

- Controller produces a control action proportional to the derivative of the error
    - anticipates direction of error changes
    - can decrease overshoot
    - can dampen oscillatory behavior
    - BUT: increases sensitivity to high frequency noise in  $e(t)$
  - Normally, derivative term used only in conjunction with P and/or I terms
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# The discrete form of the D term:

Compute the slope of  $e(t)$  at the current sample time:

$$y(nT) = K_D \left[ \frac{e(nT) - e(nT - T)}{T} \right] = \frac{K_D}{T} [e(nT) - e(nT - T)]$$

Z transform: 
$$Y(z) = \frac{K_D}{T} [E(z) - E(z)z^{-1}]$$

Transfer function: 
$$\frac{Y(z)}{E(z)} = \frac{K_D}{T} [1 - z^{-1}] = \frac{K_D}{T} \left[ \frac{z - 1}{z} \right]$$

# Implementing the PID controller

Combine the discrete P, I and D terms:

$$a(nT) = K_p e(nT) + y(nT - T) + \frac{K_I T}{2} [e(nT) + e(nT - T)] + \frac{K_D}{2} [e(nT) - e(nT - T)]$$

Consider the previous sample time:

$$a(nT - T) = K_p e(nT - T) + y(nT - T) + \frac{K_D}{2} [e(nT - T) - e(nT - 2T)]$$

Solve 2<sup>nd</sup> equation for  $y(nT - T)$  and substitute into 1<sup>st</sup> equation:

$$\begin{aligned} a(nT) = & K_p e(nT) + a(nT - T) - K_p e(nT - T) - \frac{K_D}{T} [e(nT - T) - e(nT - 2T)] \\ & + \frac{K_I T}{2} [e(nT) + e(nT - T)] + \frac{K_D}{T} [e(nT) - e(nT - T)] \end{aligned}$$

# Implementing the PID controller (2)

Simplify by combining terms involving  $e(nT)$ ,  $e(nT-T)$ ,  $e(nT-2T)$ :

$$a(nT) = a(nT - T) + A_0e(nT) - A_1e(nT - T) + A_2e(nT - 2T)$$

$A_0$ ,  $A_1$ ,  $A_2$  are constants

$e(nT)$ ,  $e(nT-T)$ ,  $e(nT-2T)$  are the 3 most recent error values

$a(nT-T)$  is the previous control action

Software procedure at each sample time:

- Sample speed and compute error  $e(nT)$
- Calculate new control action: 3 multiply, 2 add, 1 subtract
- Update duty cycle with new control action
- “Delay” error values to get ready for next sample



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# Other conversion approaches

- Can use Matlab to perform the conversion
- See the lab write up for detailed explanation



# Some practical issues

- Interrupt service routine with PID calculation time cannot exceed interrupt period
  - Pre-compute all constants ( $A_0, A_1, A_2$ )
  - Avoid floating-point (real #) operations
  - Represent fractions as ratio of integers
  - Denominator of the form  $2^k$  allows shift instead of divide

Example:  $A_0 e(nT)$ , where  $A_0 = 0.312$

$$0.312 = 312/1000 = 80/256 \text{ (approximately)}$$

$$A_0 e(nT) = 80 * e(nT) / 2^8 = (80 * e(nT)) \gg 8 \text{ (in C)}$$

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# More practical issues

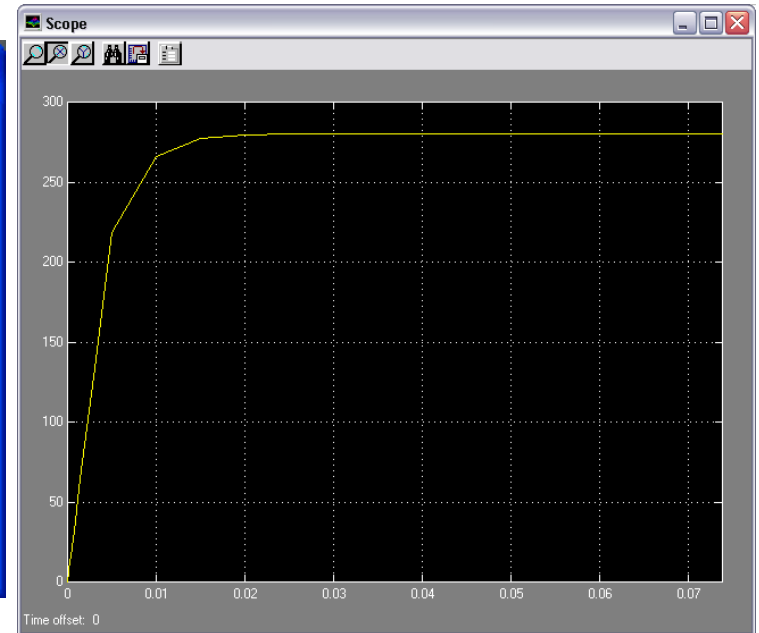
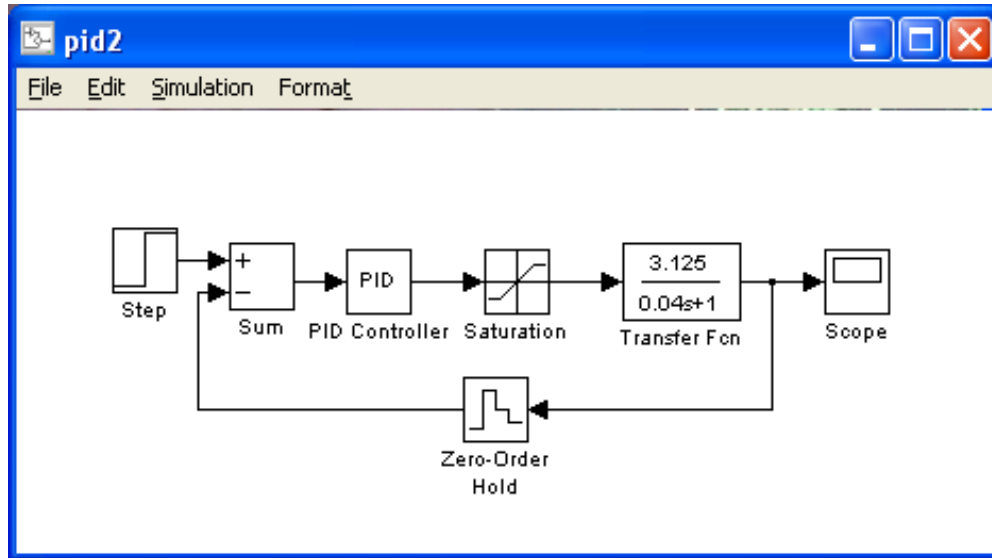
- Duty cycle cannot exceed 100%, nor go below 0%
    - Saturate values in Simulink model
  - System operation is discrete, not continuous
    - Use zero-order hold for speed in Simulink model
  - Simulink simulation gives OK starting values for constants, but real system usually varies from the model
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# Test program requirements

- Eight switch-selectable settings – stopped and seven increasing speed values.
  - Respond to speed setting changes at any time while the motor is running (without stopping the motor)
  - Respond to changes in motor load to return speed to the selected setting (we won't test this)
  - Rise/fall times at least 50% faster than the uncompensated motor
  - No steady state error
  - Minimal overshoot of the desired speed while responding to a change
  - Fast settling time after responding to a change
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# Design the controller in Matlab/Simulink



Select P-I-D constants to produce the desired response.

- Start with P value to improve response time
- Use I term to eliminate steady-state error
- Use D term to further improve response

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# Lab Procedure

- Re-verify hardware from previous labs  
**Note that circuits can still be damaged with incorrect connections/operation!**
  - Design your PID controller in Matlab/Simulink (determine the P-I-D constants)
  - Modify the software to implement the PID controller
  - Test the controller by measuring responses to step inputs
  - Compare the compensated and uncompensated step input responses
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