

## Problem Chapter I.7

The planar R-RTR-RTR mechanism considered is shown in Fig. 1. The driver link is the rigid link 1 (the link  $AB$ ). The following numerical data are given:  $AB = 0.140$  m,  $AC = 0.060$  m,  $AE = 0.250$  m,  $CD = 0.150$  m. The angle of the driver link 1 with the horizontal axis is  $\phi = 30^\circ$ . The constant angular speed of the driver link 1 is 50 rpm.

### Position analysis for an input angle

#### *Position of joint A*

A Cartesian reference frame  $xOy$  is selected. The joint  $A$  is in the origin of the reference frame, that is,  $A \equiv O$ ,

$$x_A = 0, y_A = 0. \quad (1)$$

#### *Position of joint C*

The coordinates of the joint  $C$  are

$$x_C = 0, y_C = AC = 0.060 \text{ m}. \quad (2)$$

#### *Position of joint E*

The coordinates of the joint  $E$  are

$$x_E = 0, y_E = -AE = -0.250 \text{ m}. \quad (3)$$

#### *Position of joint B*

The unknowns are the coordinates of the joint  $B$ ,  $x_B$  and  $y_B$ . Because the joint  $A$  is fixed and the angle  $\phi$  is known, the coordinates of the joint  $B$  are computed from the following expressions

$$\begin{aligned} x_B &= AB \cos \phi = 0.140 \cos 30^\circ = 0.121 \text{ m}, \\ y_B &= AB \sin \phi = 0.140 \sin 30^\circ = 0.070 \text{ m}. \end{aligned} \quad (4)$$

#### *Position of joint D*

The unknowns are the coordinates of the joint  $D$ ,  $x_D$  and  $y_D$ . The joint  $D$  is located on the line  $BC$ :

$$\begin{aligned} \frac{y_D - y_C}{x_D - x_C} &= \frac{y_B - y_C}{x_B - x_C} \quad \text{or} \\ (x_B - x_C)(y_D - y_C) &= (x_D - x_C)(y_B - y_C) \quad \text{or} \\ (0.121 - 0)(y_D - 0.060) &= (x_D - 0)(0.070 - 0.060). \end{aligned} \quad (5)$$

Furthermore, the length of the segment  $CD$  is constant

$$\begin{aligned}(x_C - x_D)^2 + (y_C - y_D)^2 &= CD^2 \quad \text{or} \\ (0 - x_D)^2 + (0.060 - y_D)^2 &= 0.150^2.\end{aligned}\quad (6)$$

The Eqs. (5) and (6) form a system from which the coordinates of the joint  $D$  can be computed. Two sets of solutions are found for the position of the joint  $D$ . These solutions are located at the intersection of the line  $BC$  with the circle centered in  $C$  and radius  $CD$  (Fig. 2), and they have the following numerical values:

$$\begin{aligned}x_{D1} &= -0.149 \text{ m}, \quad y_{D1} = 0.047 \text{ m}, \\ x_{D2} &= 0.149 \text{ m}, \quad y_{D2} = 0.072 \text{ m}.\end{aligned}$$

To determine the correct position of the joint  $D$  for the mechanism, an additional condition is needed.

For the first quadrant,  $0 \leq \phi \leq 90^\circ$ , the condition is  $x_D \leq x_C$ .

Because  $x_C = 0$  m, the coordinates of the joint  $D$  are

$$\begin{aligned}x_D &= x_{D1} = -0.149 \text{ m}, \\ y_D &= y_{D1} = 0.047 \text{ m}.\end{aligned}$$

*Angle  $\phi_2$*

The angle of link 2 (or link 3) with the horizontal axis is calculated from the slope of the straight line  $BC$ :

$$\phi_2 = \phi_3 = \arctan \frac{y_B - y_C}{x_B - x_C} = \arctan \frac{0.070 - 0.060}{0.121 - 0} = 0.082 \text{ rad} = 4.715^\circ.$$

*Angle  $\phi_4$*

The angle of link 5 (or link 4) with the horizontal axis is obtained from the slope of the straight line  $ED$ :

$$\phi_4 = \phi_5 = \arctan \frac{y_E - y_D}{x_E - x_D} = \arctan \frac{-0.250 - 0.047}{0 + 0.149} = -1.105 \text{ rad} = -63.333^\circ.$$

## Velocity and Acceleration Analysis

### Algebraic Method

The velocity of the point  $B = B_1$  on the link 1 is

$$\mathbf{v}_B = \mathbf{v}_{B_1} = \mathbf{v}_A + \boldsymbol{\omega}_1 \times \mathbf{r}_{AB} = \boldsymbol{\omega}_1 \times \mathbf{r}_B,$$

where  $\mathbf{v}_A \equiv \mathbf{0}$  is the velocity of the origin  $A \equiv O$ .

The angular velocity of link 1 is

$$\boldsymbol{\omega}_1 = \omega_1 \mathbf{k} = \frac{\pi n}{30} \mathbf{k} = \frac{\pi(50)}{30} \mathbf{k} = 5.235 \mathbf{k} \text{ rad/s.}$$

the position vector of point  $B$  is

$$\mathbf{r}_{AB} = \mathbf{r}_B - \mathbf{r}_A = \mathbf{r}_B = x_B \mathbf{i} + y_B \mathbf{j} + z_B \mathbf{k} = 0.121 \mathbf{i} + 0.070 \mathbf{j} \text{ m.}$$

The velocity of point  $B_2$  on the link 2 is  $\mathbf{v}_{B_2} = \mathbf{v}_{B_1}$  because between the links 1 and 2 there is a rotational joint.

The velocity of  $B_1 = B_2$  is

$$\mathbf{v}_{B_1} = \mathbf{v}_{B_2} = \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 0 & 0 & \omega \\ x_B & y_B & 0 \end{vmatrix} = \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 0 & 0 & 5.235 \\ 0.121 & 0.070 & 0 \end{vmatrix} = -0.366 \mathbf{i} + 0.634 \mathbf{j} \text{ m/s.}$$

The acceleration of the point  $B = B_1$  on the link 1 is

$$\begin{aligned} \mathbf{a}_B = \mathbf{a}_{B_1} = \mathbf{a}_{B_2} &= \mathbf{a}_A + \boldsymbol{\alpha}_1 \times \mathbf{r}_B + \boldsymbol{\omega}_1 \times (\boldsymbol{\omega}_1 \times \mathbf{r}_B) = \boldsymbol{\alpha}_1 \times \mathbf{r}_B - \omega_1^2 \mathbf{r}_B \\ &= -\omega_1^2 \mathbf{r}_B = -5.235^2 (0.121 \mathbf{i} + 0.070 \mathbf{j}) = -3.323 \mathbf{i} - 1.919 \mathbf{j} \text{ m/s}^2. \end{aligned}$$

The angular acceleration of link 1 is  $\boldsymbol{\alpha}_1 = \dot{\boldsymbol{\omega}}_1 = \mathbf{0}$ .

The velocity of the point  $B_3$  on the link 3 is calculated in terms of the velocity of the point  $B_2$  on the link 2

$$\mathbf{v}_{B_3} = \mathbf{v}_{B_2} + \mathbf{v}_{B_3 B_2}^{rel} = \mathbf{v}_{B_2} + \mathbf{v}_{B_{32}}, \quad (7)$$

where  $\mathbf{v}_{B_3 B_2}^{rel} = \mathbf{v}_{B_{32}}$  is the relative acceleration of  $B_3$  with respect to  $B_2$  on link 3. This relative velocity is parallel to the sliding direction  $BC$ ,  $\mathbf{v}_{B_{32}} \parallel BC$ , or

$$\mathbf{v}_{B_{32}} = v_{B_{32}} \cos \phi_2 \mathbf{i} + v_{B_{32}} \sin \phi_2 \mathbf{j}, \quad (8)$$

where  $\phi_2 = 4.715^\circ$  is known from position analysis. The points  $B_3$  and  $C$  are on the link 3 and

$$\mathbf{v}_{B_3} = \mathbf{v}_C + \boldsymbol{\omega}_3 \times \mathbf{r}_{CB} = \boldsymbol{\omega}_3 \times (\mathbf{r}_B - \mathbf{r}_C), \quad (9)$$

where  $\mathbf{v}_C \equiv \mathbf{0}$  and the angular velocity of link 3 is

$$\boldsymbol{\omega}_3 = \omega_3 \mathbf{k}.$$

Equations (7), (8), and (9) give

$$\begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 0 & 0 & \omega_3 \\ x_B - x_C & y_B - y_C & 0 \end{vmatrix} = v_{B_2} + v_{B_{32}} \cos \phi_2 \mathbf{i} + v_{B_{32}} \sin \phi_2 \mathbf{j}. \quad (10)$$

Equation (10) represents a vectorial equations with two scalar components on  $x$ -axis and  $y$ -axis and with two unknowns  $\omega_3$  and  $v_{B_{32}}$

$$\begin{aligned} -\omega_3(y_B - y_C) &= v_{B_x} + v_{B_{32}} \cos \phi_2, \\ \omega_3(x_B - x_C) &= v_{B_y} + v_{B_{32}} \sin \phi_2, \end{aligned}$$

or

$$\begin{aligned} -\omega_3(0.070 - 0.060) &= -0.366 + v_{B_{32}} \cos 4.715^\circ, \\ \omega_3(0.121 - 0) &= 0.634 + v_{B_{32}} \sin 4.715^\circ. \end{aligned}$$

It results

$$\omega_3 = \omega_2 = 5.448 \text{ rad/s} \quad \text{and} \quad v_{B_{32}} = 0.313 \text{ m/s}.$$

The acceleration of the point  $B_3$  on the link 3 is calculated in terms of the acceleration of the point  $B_2$  on the link 2

$$\mathbf{a}_{B_3} = \mathbf{a}_{B_2} + \mathbf{a}_{B_3B_2}^{rel} + \mathbf{a}_{B_3B_2}^{cor} = \mathbf{a}_{B_2} + \mathbf{a}_{B_{32}} + \mathbf{a}_{B_{32}}^{cor}, \quad (11)$$

where  $\mathbf{a}_{B_3B_2}^{rel} = \mathbf{a}_{B_{32}}$  is the relative acceleration of  $B_3$  with respect to  $B_2$  on link 3. This relative acceleration is parallel to the sliding direction  $BC$ ,  $\mathbf{a}_{B_{32}} \parallel BC$ , or

$$\mathbf{a}_{B_{32}} = a_{B_{32}} \cos \phi_2 \mathbf{i} + a_{B_{32}} \sin \phi_2 \mathbf{j}. \quad (12)$$

The Coriolis acceleration of  $B_3$  relative to  $B_2$  is

$$\begin{aligned} \mathbf{a}_{B_{32}}^{cor} &= 2 \boldsymbol{\omega}_3 \times \mathbf{v}_{B_{32}} = 2 \boldsymbol{\omega}_2 \times \mathbf{v}_{B_{32}} = 2 \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 0 & 0 & \omega_3 \\ v_{B_{32}} \cos \phi_2 & v_{B_{32}} \sin \phi_2 & 0 \end{vmatrix} = \\ &= 2(-\omega_3 v_{B_{32}} \sin \phi_2 \mathbf{i} + \omega_3 v_{B_{32}} \cos \phi_2 \mathbf{j}) = \\ &= 2[-5.448(0.313) \sin 4.715^\circ \mathbf{i} + 5.448(0.313) \cos 4.715^\circ \mathbf{j}] = \\ &= -0.280 \mathbf{i} + 3.400 \mathbf{j} \text{ m/s}^2. \end{aligned} \quad (13)$$

The points  $B_3$  and  $C$  are on the link 3 and

$$\mathbf{a}_{B_3} = \mathbf{a}_C + \boldsymbol{\alpha}_3 \times \mathbf{r}_{CB} - \omega_3^2 \mathbf{r}_{CB}, \quad (14)$$

where  $\mathbf{a}_C \equiv \mathbf{0}$  and the angular acceleration of link 3 is

$$\boldsymbol{\alpha}_3 = \alpha_3 \mathbf{k}.$$

Equations (11), (12), (13), and (14) give

$$\begin{aligned} \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 0 & 0 & \alpha_3 \\ x_B - x_C & y_B - y_C & 0 \end{vmatrix} - \omega_3^2 (\mathbf{r}_B - \mathbf{r}_C) = \\ \mathbf{a}_{B_2} + a_{B_{32}} (\cos \phi_2 \mathbf{i} + \sin \phi_2 \mathbf{j}) + 2 \boldsymbol{\omega}_3 \times \mathbf{v}_{B_{32}}. \end{aligned} \quad (15)$$

Equation (15) represents a vectorial equations with two scalar components on  $x$ -axis and  $y$ -axis and with two unknowns  $\alpha_3$  and  $a_{B_{32}}$

$$\begin{aligned} -\alpha_3 (y_B - y_C) - \omega_3^2 (x_B - x_C) &= a_{B_x} + a_{B_{32}} \cos \phi_2 - 2\omega_3 v_{B_{32}} \sin \phi_2, \\ \alpha_3 (x_B - x_C) - \omega_3^2 (y_B - y_C) &= a_{B_y} + a_{B_{32}} \sin \phi_2 + 2\omega_3 v_{B_{32}} \cos \phi_2, \end{aligned}$$

or

$$\begin{aligned} -\alpha_3 (0.070 - 0.060) - 5.448^2 (0.121 - 0) &= \\ -3.323 + a_{B_{32}} \cos 4.715^\circ - 2(5.448)(0.313) \sin 4.715^\circ, \\ \alpha_3 (0.121 - 0) - 5.448^2 (0.070 - 0.060) &= \\ -1.919 + a_{B_{32}} \sin 4.715^\circ + 2(5.448)(0.313) \cos 4.715^\circ. \end{aligned}$$

It results

$$\alpha_3 = \alpha_2 = 14.568 \text{ rad/s}^2 \text{ and } a_{B_{32}} = -0.140 \text{ m/s}^2.$$

The velocity of  $D_3$  is

$$\begin{aligned} \mathbf{v}_{D_3} = \mathbf{v}_{D_4} = \mathbf{v}_C + \boldsymbol{\omega}_3 \times \mathbf{r}_{CD} = \boldsymbol{\omega}_3 \times (\mathbf{r}_D - \mathbf{r}_C) = \\ \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 0 & 0 & \omega_3 \\ x_D - x_C & y_D - y_C & 0 \end{vmatrix} = \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 0 & 0 & 5.448 \\ -0.149 - 0 & 0.047 - 0.060 & 0 \end{vmatrix} = \\ 0.067\mathbf{i} - 0.814\mathbf{j} \text{ m/s.} \end{aligned}$$

The acceleration of  $D_3$  is

$$\begin{aligned} \mathbf{a}_{D_3} = \mathbf{a}_{D_4} = \mathbf{a}_C + \boldsymbol{\alpha}_3 \times \mathbf{r}_{CD} - \omega_3^2 \mathbf{r}_{CD} = \boldsymbol{\alpha}_3 \times (\mathbf{r}_D - \mathbf{r}_C) - \omega_3^2 (\mathbf{r}_D - \mathbf{r}_C) = \\ \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 0 & 0 & \alpha_3 \\ x_D - x_C & y_D - y_C & 0 \end{vmatrix} - \omega_3^2 [(x_D - x_C)\mathbf{i} + (y_D - y_C)\mathbf{j}] = \\ \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 0 & 0 & 14.568 \\ -0.149 - 0 & 0.047 - 0.060 & 0 \end{vmatrix} - \\ 5.448^2 [(-0.149 - 0)\mathbf{i} + (0.047 - 0.060)\mathbf{j}] = \\ 4.617\mathbf{i} - 1.811\mathbf{j} \text{ m/s}^2. \end{aligned}$$

The velocity of the point  $D_5$  on the link 5 is calculated in terms of the velocity of the point  $D_4$  on the link 4

$$\mathbf{v}_{D_5} = \mathbf{v}_{D_4} + \mathbf{v}_{D_{54}}. \quad (16)$$

This relative velocity of  $D_5$  with respect to  $D_4$  is parallel to the sliding direction  $DE$ ,  $\mathbf{v}_{D_{54}} \parallel DE$ , or

$$\mathbf{v}_{D_{54}} = v_{D_{54}} \cos \phi_5 \mathbf{i} + v_{D_{54}} \sin \phi_5 \mathbf{j}. \quad (17)$$

The points  $D_5$  and  $E$  are on the link 5 and

$$\mathbf{v}_{D_5} = \mathbf{v}_E + \boldsymbol{\omega}_5 \times \mathbf{r}_{ED} = \boldsymbol{\omega}_5 \times (\mathbf{r}_D - \mathbf{r}_E), \quad (18)$$

where  $\mathbf{v}_E \equiv \mathbf{0}$  and the angular velocity of link 5 is

$$\boldsymbol{\omega}_5 = \omega_5 \mathbf{k}.$$

Equations (16), (17), and (18) give

$$\begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 0 & 0 & \omega_5 \\ x_D - x_E & y_D - y_E & 0 \end{vmatrix} = \mathbf{v}_{D_4} + v_{D_{54}} (\cos \phi_5 \mathbf{i} + \sin \phi_5 \mathbf{j}). \quad (19)$$

Equation (19) represents a vectorial equations with two scalar components on  $x$ -axis and  $y$ -axis and with two unknowns  $\omega_5$  and  $v_{D_{54}}$

$$\begin{aligned} -\omega_5(y_D - y_E) &= v_{D_{4x}} + v_{D_{54}} \cos \phi_5, \\ \omega_5(x_D - x_E) &= v_{D_{4y}} + v_{D_{54}} \sin \phi_5, \end{aligned}$$

or

$$\begin{aligned} -\omega_5(0.047 - 0.250) &= 0.067 + v_{D_{54}} \cos(-63.333^\circ), \\ \omega_5(-0.149 - 0) &= -0.814 + v_{D_{54}} \sin(-63.333^\circ). \end{aligned}$$

It results

$$\omega_5 = \omega_4 = 0.917 \text{ rad/s} \quad \text{and} \quad v_{D_{54}} = -0.757 \text{ m/s}.$$

The acceleration of the point  $D_5$  on the link 5 is calculated in terms of the acceleration of the point  $D_4$  on the link 4

$$\mathbf{a}_{D_5} = \mathbf{a}_{D_4} + \mathbf{a}_{D_{54}} + \mathbf{a}_{D_{54}}^{cor}, \quad (20)$$

This relative acceleration  $\mathbf{a}_{B_{32}}$  is parallel to the sliding direction  $DE$ ,  $\mathbf{a}_{D_{54}} \parallel DE$ , or

$$\mathbf{a}_{D_{54}} = a_{D_{54}} \cos \phi_5 \mathbf{i} + a_{D_{54}} \sin \phi_5 \mathbf{j}. \quad (21)$$

The Coriolis acceleration of  $D_5$  relative to  $D_4$  is

$$\begin{aligned} \mathbf{a}_{D_{54}}^{cor} &= 2 \boldsymbol{\omega}_4 \times \mathbf{v}_{D_{54}} = 2 \boldsymbol{\omega}_5 \times \mathbf{v}_{D_{54}} = 2 \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 0 & 0 & \omega_5 \\ v_{D_{54}} \cos \phi_5 & v_{D_{54}} \sin \phi_5 & 0 \end{vmatrix} = \\ &= 2(-\omega_5 v_{D_{54}} \sin \phi_5 \mathbf{i} + \omega_5 v_{D_{54}} \cos \phi_5 \mathbf{j}) = \\ &= 2[-0.917(-0.757) \sin(-63.333^\circ) \mathbf{i} + 0.917(-0.757) \cos(-63.333^\circ) \mathbf{j}] = \\ &= -0.280 \mathbf{i} + 3.400 \mathbf{j} \text{ m/s}^2. \end{aligned} \quad (22)$$

The points  $D_5$  and  $E$  are on the link 5 and

$$\mathbf{a}_{D_5} = \mathbf{a}_E + \boldsymbol{\alpha}_5 \times \mathbf{r}_{ED} - \omega_5^2 \mathbf{r}_{ED}, \quad (23)$$

where  $\mathbf{a}_E \equiv \mathbf{0}$  and the angular acceleration of link 5 is

$$\boldsymbol{\alpha}_5 = \alpha_5 \mathbf{k}.$$

Equations (20), (21), (22), and (23) give

$$\begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 0 & 0 & \alpha_5 \\ x_D - x_E & y_D - y_E & 0 \end{vmatrix} - \omega_5^2(\mathbf{r}_D - \mathbf{r}_E) = \mathbf{a}_{D_4} + a_{D_{54}}(\cos \phi_5 \mathbf{i} + \sin \phi_5 \mathbf{j}) + 2 \boldsymbol{\omega}_5 \times \mathbf{v}_{D_{54}}. \quad (24)$$

Equation (24) represents a vectorial equations with two scalar components on  $x$ -axis and  $y$ -axis and with two unknowns  $\alpha_5$  and  $a_{D_{54}}$

$$\begin{aligned} -\alpha_5(y_D - y_E) - \omega_5^2(x_D - x_E) &= a_{D_{4x}} + a_{D_{54}} \cos \phi_5 - 2\omega_5 v_{D_{54}} \sin \phi_5, \\ \alpha_5(x_D - x_E) - 2\omega_5^2(y_D - y_E) &= a_{D_{4y}} + a_{D_{54}} \sin \phi_5 + 2\omega_5 v_{D_{54}} \cos \phi_5, \end{aligned}$$

or

$$\begin{aligned} -\alpha_5(0.047 - 0.250) - 0.917^2(-0.149 - 0) &= \\ 4.617 + a_{D_{54}} \cos(-63.333^\circ) - 2(0.917)(-0.757) \sin(-63.333^\circ), \\ \alpha_5(-0.149 - 0) - 0.917^2(0.047 - 0.250) &= \\ -1.811 + a_{D_{54}} \sin(-63.333^\circ) + 2(0.917)(-0.757) \cos(-63.333^\circ). \end{aligned}$$

It results

$$\alpha_5 = \alpha_4 = -5.771 \text{ rad/s}^2 \quad \text{and} \quad a_{D_{54}} = 3.411 \text{ m/s}^2.$$

The *Mathematica*<sup>TM</sup> program for the velocity and acceleration analysis using the algebraic method is given in Program 4-II.

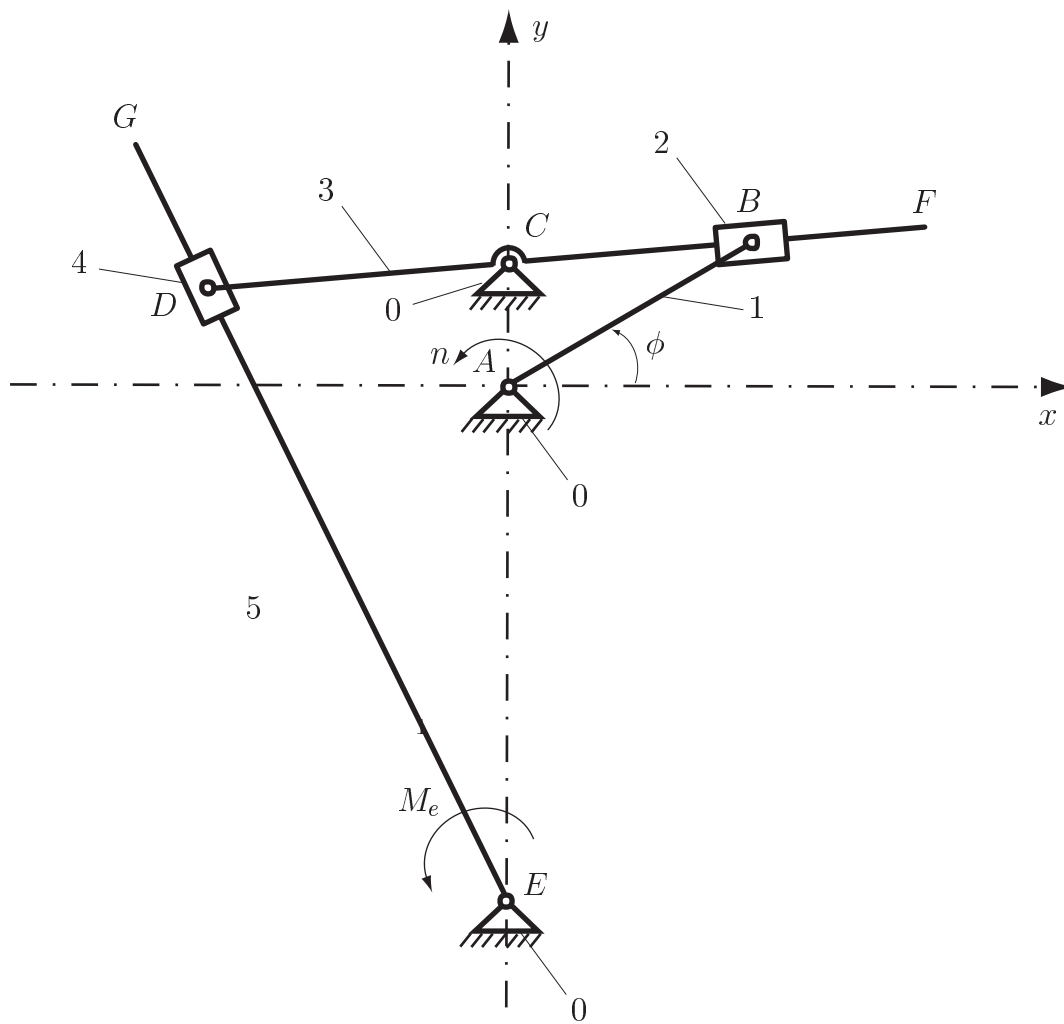


Fig. 1

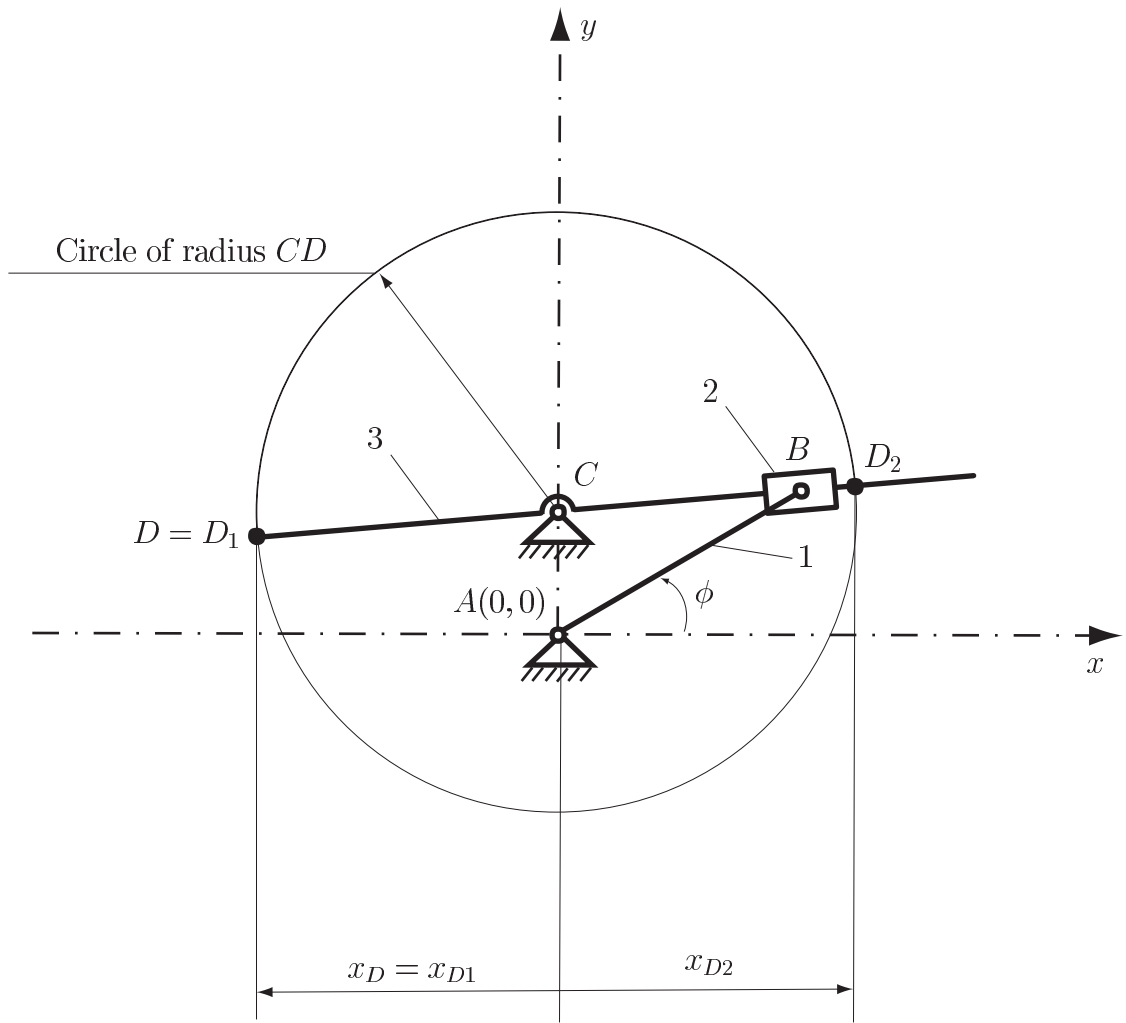


Fig. 2

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(* VELOCITY AND ACCELERATION ANALYSIS *)
(* Algebraic Method *)

Apply [Clear, Names["Global`*"] ] ;
Off[General::spell];
Off[General::spell1];

AB=0.14;
AC=0.06;
AE=0.25;
CD=0.15;
φ=π/6.;

xA = yA = 0;
rA = { xA, yA, 0 } ;

xC = 0 ;
yC = AC ;
rC = { xC, yC, 0 } ;

xE = 0 ;
yE = -AE ;
rE = { xE, yE, 0 } ;

xB = AB Cos[φ];
yB = AB Sin[φ];
rB = { xB, yB, 0 } ;

eqnD1 = ( xDs - xC )^2 + ( yDs - yC )^2 - CD^2 == 0 ;
eqnD2 = ( yB - yC )( xDs - xC ) == ( yDs - yC )( xB - xC ) ;
solutionD = Solve [ { eqnD1 , eqnD2 } , { xDs, yDs } ] ;
xD1 = xDs /. solutionD[[1]];
yD1 = yDs /. solutionD[[1]];
xD2 = xDs /. solutionD[[2]];
yD2 = yDs /. solutionD[[2]];
If [ xD1 <= xC , xD = xD1 ; yD = yD1 , xD = xD2 ; yD=yD2 ] ;
rD = { xD, yD, 0 } ;

φ2=φ3=ArcTan[(yB-yC)/(xB-xC)];
φ4=φ5=ArcTan[(yD-yE)/(xD-xE)];

markers = Table [ {
    Point [ { xA , yA } ] ,
    Point [ { xB , yB } ] ,
    Point [ { xC , yC } ] ,
    Point [ { xD , yD } ] ,
    Point [ { xE , yE } ]
} ] ;

name = Table [ {
    Text [ "A" , {0 , 0 } , { -1 , 1 } ] ,
    Text [ "B" , {xB , yB } , { 0 , -1 } ] ,
    Text [ "C" , {xC , yC } , { -1, -1 } ] ,
    Text [ "D" , {xD , yD } , { 0 , -1 } ] ,
    Text [ "E" , {xE , yE } , { -1, 1 } ]
} ] ;

graph = Graphics [
    { { RGBColor [ 1 , 0 , 0 ] ,
      Line [ { {xA,yA},{xB,yB} } ] } ,
      { RGBColor [ 0 , 1 , 0 ] ,
      Line [ { {xB,yB} , {xD,yD} } ] } ,
      { RGBColor [ 0 , 0 , 1 ] ,

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Line [ { {xD,yD}, {xE,yE} } ] ,
{ RGBColor [ 1 , 1 , 1 ] ,
  PointSize [ 0.01 ] , markers } ,
{ name } } ] ;

Show [ Graphics [ graph ] ,
  PlotRange -> { { -.25 , .25 } ,
                { -.3 , .25 } } ,
  Frame -> True,
  AxesOrigin -> {xA,yA},
  FrameLabel -> {"x","y"},
  Axes -> {True,True},
  AspectRatio -> Automatic ] ;

n = 50.; (* rpm *)
ω = ω1 = { 0, 0, n π/30 } ;
α = α1 = D[ω,t] ;
Print["ω = ω1 = ",ω," rad/s"];
Print["α = α1 = ",α," rad/s^2"];

vB=vB1=vB2=Cross[ω1,rB];
Print["vB = vB1 = ω1 x rB = ",vB," m/s"];
Print["vB1 = vB2 "];

aB=aB1=aB2=Cross[α1,rB]-ω1.ω1 rB;
Print["aB = aB1 = α1 x rB - ω1^2 rB = ",aB," m/s^2"];
Print["aB1 = aB2 "];

(*--ω3--*)

ω3u={ 0, 0, ω3z};
Print["ω3 = ω2 = ",ω3u];

vB3u=Cross[ω3u,rB-rC];
Print["vB3 = vC + ω3 x (rB-rC)"];

vB32u={ v32 Cos[φ2], v32 Sin[φ2], 0};
Print["vB32={vB32 Cos[φ2],vB32 Sin[φ2],0}"];

Print["vB3 = vB2 + vB32 => ω3z, vB32"];

eqvB=vB3u-vB2-vB32u;
solutionvB=Solve[{eqvB[[1]]==0,eqvB[[2]]==0},{ω3z,v32}];
ω3=ω2=ω3u/.solutionvB[[1]];
vB32=vB32u/.solutionvB[[1]];
Print["ω3 = ω2 = ",ω3," rad/s"];
Print["vB32 = ",v32/.solutionvB[[1]]," m/s"];
Print["vB32v = ",vB32," m/s"];

vB3=vB3u/.solutionvB[[1]];
Print["vB3 = ",vB3," m/s"];

vD3=vD4=Cross[ω3,rD-rC];
Print["vD3 = vD4 = vC + ω3 x (rD-rC) = ",vD3," m/s"];

(*--α3--*)

α3u={ 0, 0, α3z};
Print["α3 = α2 = ",α3u];

aB3u=Cross[α3u,rB-rC]-ω3.ω3(rB-rC) ;
Print["aB3 = aC + α3 x (rB-rC) - ω3.ω3(rB-rC)"];

```

```

aB32u={ a32 Cos[φ2], a32 Sin[φ2], 0};
Print["aB32={aB32 Cos[φ2],aB32 Sin[φ2],0}"];

aB32cor = 2 Cross[ω3,vB32];
Print["aB32cor = 2 ω3 x vB32 = ",aB32cor," m/s^2"];

Print["aB3 = aB2 + aB32 + aB32cor => α3z, aB32"];

eqaB=aB3u-aB2-aB32u-aB32cor;
solutionaB=Solve[{eqaB[[1]]==0,eqaB[[2]]==0},{α3z,a32}];
α3=α2=α3u/.solutionaB[[1]];
aB32=aB32u/.solutionaB[[1]];
Print["α3 = α2 = ",α3," rad/s^2"];
Print["aB32 = ",a32/.solutionaB[[1]]," m/s^2"];
Print["aB32v = ",aB32," m/s^2"];

aB3=aB3u/.solutionaB[[1]];
Print["aB3 = ",aB3," m/s^2"];

aD3=aD4=Cross[α3,rD-rC]-ω3.ω3(rD-rC) ;
Print["aD3 = aD4 = aC + α3 x (rD-rC) - ω3.ω3(rD-rC)"];
Print["aD3 = ",aD3," m/s^2"];

(*--ω5--*)

ω5u={ 0, 0, ω5z};
Print["ω5 = ω5 = ",ω5u];

vD5u=Cross[ω5u,rD-rE];
Print["vD5 = vE + ω5 x (rD-rE)"];

vD54u={ v54 Cos[φ4], v54 Sin[φ4], 0};
Print["vD54={vD54 Cos[φ4],vD54 Sin[φ4],0}"];

Print["vD5 = vD4 + vD54 => ω5z, vD54"];

eqvD=vD5u-vD4-vD54u;
solutionvD=Solve[{eqvD[[1]]==0,eqvD[[2]]==0},{ω5z,v54}];
ω5=ω4=ω5u/.solutionvD[[1]];
vD54=vD54u/.solutionvD[[1]];
Print["ω5 = ω4 = ",ω5," rad/s"];
Print["vD54 = ",v54/.solutionvD[[1]]," m/s"];
Print["vD54v = ",vD54," m/s"];

vD5=vD5u/.solutionvD[[1]];
Print["vD5 = ",vD5," m/s"];

(*--α5--*)

α5u={ 0, 0, α5z};
Print["α5 = α4 = ",α5u];

aD5u=Cross[α5u,rD-rE]-ω5.ω5(rD-rE) ;
Print["aD5 = aE + α5 x (rD-rE) - ω5.ω5(rD-rE)"];

aD54u={ a54 Cos[φ5], a54 Sin[φ5], 0};
Print["aD54={aD54 Cos[φ5],aD54 Sin[φ5],0}"];

aD54cor = 2 Cross[ω5,vD54];
Print["aD54cor = 2 ω5 x vD54 = ",aD54cor," m/s^2"];

Print["aD5 = aD4 + aD54 + aD54cor => α5z, aD54"];

eqaD=aD5u-aD4-aD54u-aD54cor;
solutionaD=Solve[{eqaD[[1]]==0,eqaD[[2]]==0},{α5z,a54}];

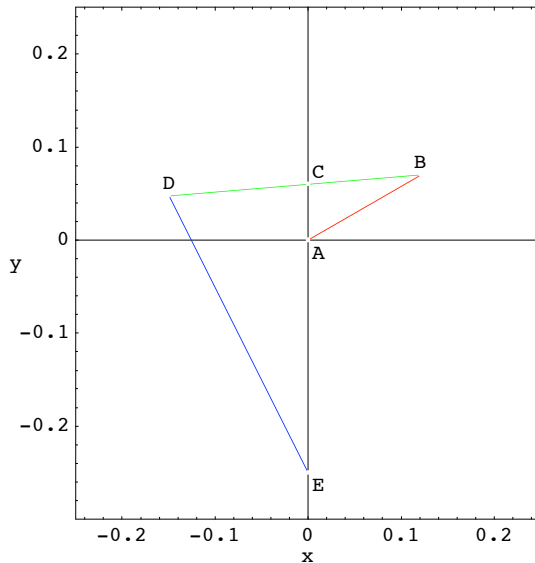
```

```

α5=α4=α5u/.solutionaD[[1]];
aD54=aD54u/.solutionaD[[1]];
Print["α5 = α4 = ",α5," rad/s^2"];
Print["aD54 = ",a54/.solutionaD[[1]]," m/s^2"];
Print["aD54v = ",aD54," m/s^2"];

aD5=aD5u/.solutionaD[[1]];
Print["aD5 = ",aD5," m/s^2"];

```



```

ω = ω1 = {0, 0, 5.23599} rad/s

α = α1 = {0, 0, 0} rad/s^2

vB = vB1 = ω1 x rB = {-0.366519, 0.63483, 0.} m/s

vB1 = vB2

aB = aB1 = α1 x rB - ω1^2 rB = {-3.32396, -1.91909, 0.} m/s^2

aB1 = aB2

ω3 = ω2 = {0, 0, ω3z}

vB3 = vC + ω3 x (rB-rC)

vB32={vB32 Cos[φ2],vB32 Sin[φ2],0}

vB3 = vB2 + vB32 => ω3z, vB32

ω3 = ω2 = {0, 0, 5.44826} rad/s

vB32 = 0.313096 m/s

vB32v = {0.312037, 0.0257363, 0} m/s

vB3 = {-0.0544826, 0.660566, 0.} m/s

vD3 = vD4 = vC + ω3 x (rD-rC) = {0.0671766, -0.814473, 0.} m/s

α3 = α2 = {0, 0, α3z}

```

```
aB3 = aC +  $\alpha_3$  x (rB-rC) -  $\omega_3.\omega_3$ (rB-rC)
aB32={aB32 Cos[ $\phi_2$ ],aB32 Sin[ $\phi_2$ ],0}
aB32cor = 2  $\omega_3$  x vB32 = {-0.280436, 3.40011, 0.} m/s^2
aB3 = aB2 + aB32 + aB32cor =>  $\alpha_3z$ , aB32
 $\alpha_3 = \alpha_2 = \{0, 0, 14.5681\}$  rad/s^2
aB32 = -0.140694 m/s^2
aB32v = {-0.140218, -0.011565, 0} m/s^2
aB3 = {-3.74462, 1.46946, 0.} m/s^2
aD3 = aD4 = aC +  $\alpha_3$  x (rD-rC) -  $\omega_3.\omega_3$ (rD-rC)
aD3 = {4.61708, -1.81183, 0.} m/s^2
 $\omega_5 = \omega_5 = \{0, 0, \omega_5z\}$ 
vD5 = vE +  $\omega_5$  x (rD-rE)
vD54={vD54 Cos[ $\phi_4$ ],vD54 Sin[ $\phi_4$ ],0}
vD5 = vD4 + vD54 =>  $\omega_5z$ , vD54
 $\omega_5 = \omega_4 = \{0, 0, 0.917134\}$  rad/s
vD54 = -0.757991 m/s
vD54v = {-0.34018, 0.677368, 0} m/s
vD5 = {-0.273003, -0.137105, 0.} m/s
 $\alpha_5 = \alpha_4 = \{0, 0, \alpha_5z\}$ 
aD5 = aE +  $\alpha_5$  x (rD-rE) -  $\omega_5.\omega_5$ (rD-rE)
aD54={aD54 Cos[ $\phi_5$ ],aD54 Sin[ $\phi_5$ ],0}
aD54cor = 2  $\omega_5$  x vD54 = {-1.24248, -0.623982, 0.} m/s^2
aD5 = aD4 + aD54 + aD54cor =>  $\alpha_5z$ , aD54
 $\alpha_5 = \alpha_4 = \{0, 0, -5.77155\}$  rad/s^2
aD54 = -3.41104 m/s^2
aD54v = {-1.53085, 3.04823, 0} m/s^2
aD5 = {1.84376, 0.612421, 0.} m/s^2
```