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Simulation of Kinematic Chains with Mathematica\textsuperscript{TM}

A planar mechanism will be analyzed and simulated using the Mathematica\textsuperscript{TM} software. The planar R-RTR-RTR mechanism considered is shown in Fig. 7.1. The driver link is the rigid link 1 (the link $AB$). The following numerical data are given: $AB = 0.140$ m, $AC = 0.060$ m, $AE = 0.250$ m, $CD = 0.150$ m. The angle of the driver link 1 with the horizontal axis $\phi$.

7.1 Position Analysis

The Mathematica\textsuperscript{TM} commands for the input data are

\begin{align*}
AB &= 0.140; \\
AC &= 0.060; \\
AE &= 0.250; \\
CD &= 0.150;
\end{align*}

Position analysis for an input angle

The angle of the driver link 1 with the horizontal axis $\phi = 30^\circ$. The Mathematica\textsuperscript{TM} command for the input angle is

\begin{equation}
\text{phi} = \text{N[Pi]/6};
\end{equation}

where \text{N[expr]} gives the numerical value of \text{expr} and Pi is the constant $\pi$, with numerical value approximately equal to 3.14159.

Position of joint $A$

A Cartesian reference frame $xOy$ is selected. The joint $A$ is in the origin of the reference frame, that is, $A \equiv O$,

\begin{equation}
\begin{aligned}
x_A &= 0, \\
y_A &= 0.
\end{aligned}
\end{equation}

(7.1)

Position of joint $C$

The coordinates of the joint $C$ are

\begin{equation}
\begin{aligned}
x_C &= 0, \\
y_C &= AC = 0.060 \text{ m}.
\end{aligned}
\end{equation}

(7.2)

Position of joint $E$

The coordinates of the joint $E$ are

\begin{equation}
\begin{aligned}
x_E &= 0, \\
y_E &= -AE = -0.250 \text{ m}.
\end{aligned}
\end{equation}

(7.3)
The Mathematica\textsuperscript{TM} commands for Eqs. (7.1), (7.2), and (7.3) are

\begin{verbatim}
xA=0; yA=0;
xC=0; yC=AC;
xE=0; yE=-AE;
\end{verbatim}

**Position of joint B**
The unknowns are the coordinates of the joint B, \( x_B \) and \( y_B \). Because the joint A is fixed and the angle \( \phi \) is known, the coordinates of the joint B are computed from the following expressions

\[
\begin{align*}
x_B &= AB \cos \phi = 0.140 \cos 30^\circ = 0.121 \text{ m}, \\
y_B &= AB \sin \phi = 0.140 \sin 30^\circ = 0.070 \text{ m}. 
\end{align*}
\]

(7.4)
The Mathematica\textsuperscript{TM} commands for Eq. (7.4) are

\begin{verbatim}
xB=AB*Cos[phi];
yB=AB*Sin[phi];
\end{verbatim}

where \texttt{phi} is the angle \( \phi \) in radians.

**Position of joint D**
The unknowns are the coordinates of the joint D, \( x_D \) and \( y_D \). Knowing the positions of the joints B and C, one can compute the slope \( m \) and the intercept \( b \) of the line \( BC \)

\[
\begin{align*}
m &= \frac{(y_B - y_C)}{(x_B - x_C)}, \\
b &= y_B - m \cdot x_B. 
\end{align*}
\]

(7.5)
The Mathematica\textsuperscript{TM} commands for Eq. (7.5) are

\begin{verbatim}
m=(yB-yC)/(xB-xC);
b=yB-m*xB;
\end{verbatim}

The joint \( D \) is located on the line \( BC \)

\[
y_D - m \cdot x_D - b = 0.
\]

(7.6)
Furthermore, the length of the segment \( CD \) is constant

\[
(x_C - x_D)^2 + (y_C - y_D)^2 = CD^2.
\]

(7.7)
The Eqs. (7.6) and (7.7) with Mathematica commands are

\[
eqnD1 = (xDsol - xC)^2 + (yDsol - yC)^2 - CD^2 = 0;  
eqnD2 = yDsol - m \cdot xDsol - b = 0;  
\]

The Eqs. (7.6) and (7.7) form a system from which the coordinates of the joint D can be computed. To solve the system of equations, a specific Mathematica command will be used. The command `Solve[eqns, vars]` attempts to solve an equation or set of equations `eqns` for the variables `vars`. For the mechanism

\[
solutionD = \text{Solve}\{\{\text{eqnD1}, \text{eqnD2}\}, \{xDsol, yDsol\}\};  
\]

Two sets of solutions are found for the position of the joint D

\[
xD1 = xDsol/.\text{solutionD}[[1]];  
yD1 = yDsol/.\text{solutionD}[[1]];  
xD2 = xDsol/.\text{solutionD}[[2]];  
yD2 = yDsol/.\text{solutionD}[[2]];  
\]

These solutions are located at the intersection of the line BC with the circle centered in C and radius CD (Fig. 7.2) and they have the following numerical values

\[
x_D1 = -0.149 \, \text{m}, \quad y_D1 = 0.047 \, \text{m},  
x_D2 = 0.149 \, \text{m}, \quad y_D2 = 0.072 \, \text{m}.  
\]

To determine the correct position of the joint D for the mechanism, an additional condition is needed. For the first quadrant, \(0 \leq \phi \leq 90^\circ\), the condition is \(x_D \leq x_C\). This condition with Mathematica is

\[
\text{If}[\text{condition}, \text{t}, \text{f}],  
\]

that gives \(\text{t}\) if \(\text{condition}\) evaluates to True, and \(\text{f}\) if it evaluates to False.

For the considered mechanism

\[
\text{If}[xD1\leqxC, xD=xD1; yD=yD1, xD=xD2; yD=yD2];  
\]
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Because $x_C = 0$ m, the coordinates of the joint $D$ are

$$x_D = x_{D1} = -0.149 \text{ m},$$

$$y_D = y_{D1} = 0.047 \text{ m}.$$  

The numerical solutions for $B$ and $D$ are printed using Mathematica

\begin{verbatim}
Print["xB = ", xB, " m "];
Print["yB = ", yB, " m "];
Print["xD = ", xD, " m "];
Print["yD = ", yD, " m "];
\end{verbatim}

The Mathematica program for the input angle $\phi = 30^\circ$ is given in Program 7.1. The program contains at the end the commands to draw the mechanism.

**Position analysis for a complete rotation**

For a complete rotation of the driver link $AB$, $0 \leq \phi \leq 360^\circ$, a step angle of $\phi = 60^\circ$ is selected.

To calculate the position analysis for a complete cycle one can use the Mathematica command `For[start, test, incr, body]`. It executes `start`, then repeatedly evaluates `body` and `incr` until `test` fails to give True. In the case of the mechanism

\begin{verbatim}
For[phi=0, phi<=2*N[Pi], phi+=N[Pi]/3, Program block];
\end{verbatim}

**Method I**

Method I uses constraint conditions for the mechanism for each quadrant. For the mechanism, there are several conditions for the position of the joint $D$.

For the angle $\phi$ located in the first quadrant $0^\circ \leq \phi \leq 90^\circ$ (Fig. 7.2), and the fourth quadrant $270^\circ \leq \phi \leq 360^\circ$ (Fig. 7.5), the following relation exists between $x_D$ and $x_C$

$$x_D \leq x_C.$$  

For the angle $\phi$ located in the second quadrant $90^\circ < \phi \leq 180^\circ$ (Fig. 7.3), and the third quadrant $180^\circ < \phi < 270^\circ$ (Fig. 7.4), the following relation
exists between $x_D$ and $x_C$

$$x_D \geq x_C.$$  

The following Mathematica$^\text{TM}$ commands are used to determine the correct position of the joint $D$ for all four quadrants

```mathematica
If[0 <= phi <= N[Pi]/2 || 3 N[Pi]/2 <= phi <= 2 N[Pi],
   If[xD1<=xC, xD=xD1; yD=yD1, xD=xD2; yD=yD2],
   If[xD1>=xC, xD=xD1; yD=yD1, xD=xD2; yD=yD2]];
```

where $|$ is the logical OR function.

The Mathematica$^\text{TM}$ program for a complete rotation of the driver link using the method I is given in Program 7.2. The graph of the mechanism for a complete rotation of the driver link is given in Fig. 7.6.

**Method II**

Another position analysis method for a complete rotation of the driver link uses constraint conditions for the initial value of the angle $\phi$. For the mechanism, the correct position of the joint $D$ is calculated using a simple function, the euclidian distance between two points $P$ and $Q$

$$d = \sqrt{(x_P - x_Q)^2 + (y_P - y_Q)^2}. \quad (7.8)$$

In Mathematica$^\text{TM}$, the following function is introduced

```mathematica
Dist[xP_, yP_, xQ_, yQ_] := Sqrt[(xP-xQ)^2 + (yP-yQ)^2];
```

For the initial angle $\phi = 0^\circ$, the constraint is $x_D \leq x_C$, so the first position of the joint $D$, that is, $D_0$, is calculated for the first step $D = D_0 = D_k$, $k = 0$. For the next position of the joint, $D_{k+1}$, there are two solutions $D^I_{k+1}$ and $D^H_{k+1}$, $k = 0, 1, 2, ...$. In order to choose the correct solution of the joint, $D_{k+1}$, it is compared the distances between the old position, $D_k$, and each new calculated positions $D^I_{k+1}$ and $D^H_{k+1}$. The distances between the known solution $D_k$ and the new solutions $D^I_{k+1}$ and $D^H_{k+1}$ are $d^I_k$ and $d^H_k$. If the distance to the first solution is less than the distance to the second solution, $d^I_k < d^H_k$, then the correct answer is $D_{k+1} = D^I_{k+1}$, else $D_{k+1} = D^H_{k+1}$ (Fig. 7.7).
The following *Mathematica*\textsuperscript{TM} commands are used to determine the correct position of the joint $D$ using a single condition for all four quadrants:

\begin{verbatim}
increment=0;
For[phi=0, phi<=2*N[Pi], phi+=N[Pi]/3,
...
If[increment==0, If[xD1<xC, xD=xD1; yD=yD1, xD=xD2; yD=yD2],
dist1=Dist[xD1,yD1,xDold,yDold];
dist2=Dist[xD2,yD2,xDold,yDold];
If[dist1<dist2, xD=xD1; yD=yD1, xD=xD2; yD=yD2 ]
];
xDold=xD;
yDold=yD;
increment++;
...
];
\end{verbatim}

Using this algorithm the correct solution is selected using just one constraint relation for the initial step and then automatically the problem is solved. In this way it is not necessary to have different constraints for different quadrants.

The *Mathematica*\textsuperscript{TM} program for a complete rotation of the driver link using the second method is given in Program 7.3.

### 7.2 Velocity and Acceleration Analysis

For the considered mechanism (Fig. 7.1) the driver link 1 is rotating with a constant speed of $n = 50$ rpm. Next a *Mathematica*\textsuperscript{TM} program for velocity and acceleration analysis will be presented.

The *Mathematica*\textsuperscript{TM} commands for the angular speed, in rad/s, are

\begin{verbatim}
n=50; (* rpm *)
omega=n*N[Pi]/30; (* rad/s *)
\end{verbatim}

The *Mathematica*\textsuperscript{TM} commands for coordinates of the joints $A$, $C$ and $E$ are

\begin{verbatim}
xA=0; yA=0;
xC=0; yC=AC;
\end{verbatim}
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\[ \text{x}_E = 0; \text{y}_E = -\text{AE}; \]

The coordinates of the joint \( B (B = B_1 = B_2 \text{ on the link 1 or 2}) \) are

\[ x_B(t) = AB \cos \phi(t) \quad \text{and} \quad y_B(t) = AB \sin \phi(t). \]

To calculate symbolically the position of the joint \( B \), the following Mathematica™ commands are used

\[
\begin{align*}
\text{x}_B &= AB \cos[\phi[t]]; \\
\text{y}_B &= AB \sin[\phi[t]]; \\
\end{align*}
\]

where \( \phi[t] \) represents the mathematical function \( \phi(t) \). The function name is \( \phi \) and it has one argument the time \( t \).

To calculate numerically the position of the joint \( B \), the symbolical variables needs to be substituted with the input data. To apply a transformation rule to a particular expression \( \text{expr} \), type \( \text{expr} /. \text{lhs} \rightarrow \text{rhs} \). To apply a sequence of rules on each part of the expression \( \text{expr} \), type \( \text{expr} /. \{ \text{lhs1} \rightarrow \text{rhs1}, \text{lhs2} \rightarrow \text{rhs2}, \ldots \} \).

For the mechanism, the transformation rule represents the initial data of the mechanism

\[ \text{initdata} = \{ \text{AB} \rightarrow 0.14, \text{AC} \rightarrow 0.06, \text{AE} \rightarrow 0.25, \text{CD} \rightarrow 0.15, \phi[t] \rightarrow N[\pi]/6, \phi'[t] \rightarrow \text{omega}, \phi''[t] \rightarrow 0 \}; \]

where \( \phi'[t] \) is the first derivative of \( \phi \) with respect to \( t \), and \( \phi''[t] \) is the second derivative of the function.

The command \( \text{Print[expr1, expr2, \ldots]} \) prints the \( \text{expr1, expr2, \ldots,} \) followed by a new line. To print the solutions of the position vector, the following commands are used

\[
\begin{align*}
\text{Print["x}_B = ", \text{x}_B," = ", \text{x}_B/.\text{initdata}, " \text{ m} \]}; \\
\text{Print["y}_B = ", \text{y}_B," = ", \text{y}_B/.\text{initdata}, " \text{ m} \]}; \\
\end{align*}
\]

The linear velocity vector of the joint \( B (B = B_1 = B_2) \) is

\[ \text{v}_B = \text{v}_{B_1} = \text{v}_{B_2} = \dot{x}_B + \dot{y}_B; \]
where
\[ \dot{x}_B = \frac{dx_B}{dt} = -AB\dot{\phi}\sin{\phi} \quad \text{and} \quad \dot{y}_B = \frac{dy_B}{dt} = AB\dot{\phi}\cos{\phi}, \]
are the components of the velocity vector of \( B \).
To calculate symbolically the components of the velocity vector using the \textit{Mathematica}\textsuperscript{TM} the command \( \text{D}[f, t] \) is used, which gives the derivative of \( f \) with respect to \( t \)
\begin{align*}
v_{Bx} &= \text{D}[x_B, t]; \\
v_{By} &= \text{D}[y_B, t];
\end{align*}
For the mechanism, \( \dot{\phi} = \omega = \frac{\pi n}{30} = \frac{\pi(50)}{30} \text{ rad/s} = 5.235 \text{ rad/s} \), and the numerical values are
\begin{align*}
\dot{x}_B &= -0.140 (5.235) \sin 30^\circ = -0.366 \text{ m/s}, \\
\dot{y}_B &= 0.140 (5.235) \cos 30^\circ = 0.634 \text{ m/s}.
\end{align*}
The solutions can be printed using \textit{Mathematica}\textsuperscript{TM}
\begin{align*}
\text{Print["vBx = ", vBx," = ", vBx/.initdata, " m/s" ]}; \\
\text{Print["vBy = ", vBy," = ", vBy/.initdata, " m/s" ]};
\end{align*}
The linear acceleration vector of the joint \( B \) \((B = B_1 = B_2)\) is
\[ a_B = \ddot{x}_B + \ddot{y}_B, \]
where
\begin{align*}
\ddot{x}_B &= \frac{d\dot{x}_B}{dt} = -AB\ddot{\phi}\cos{\phi} - AB\ddot{\phi}\sin{\phi}, \\
\ddot{y}_B &= \frac{d\dot{y}_B}{dt} = -AB\ddot{\phi}\sin{\phi} + AB\ddot{\phi}\cos{\phi},
\end{align*}
are the components of the acceleration vector of the joint \( B \).
The \textit{Mathematica}\textsuperscript{TM} commands used to calculate symbolically the components of the acceleration vector are
\[ a_{Bx} = \text{D}[v_{Bx}, t]; \]
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aBy=D[vBy, t];

For the considered mechanism the angular acceleration of the link 1 is $\ddot{\phi} = \dot{\omega} = 0$. The numerical values of the acceleration of $B$ are

\[
\ddot{x}_B = -0.140 (5.235)^2 \cos 30^\circ = -3.323 \text{ m/s}^2,
\]
\[
\ddot{y}_B = -0.140 (5.235)^2 \sin 30^\circ = -1.919 \text{ m/s}^2.
\]

The solutions printed with Mathematica™ are

\[
\text{Print["aBx = ", aBx," = ", aBx/.initdata, " m/s}^2" ]};
\]
\[
\text{Print["aBy = ", aBy," = ", aBy/.initdata, " m/s}^2" ]};
\]

The coordinates of the joint $D$ are $x_D$ and $y_D$. The Mathematica™ commands used to calculate the position of $D$ are

\[
mBC=(yB-yC)/(xB-xC);
bBC=yB-mBC \times B;
\]
\[
eqnD1=(xDsol-xC)^2+(yDsol-yC)^2-CD^2==0;
eqnD2=yDsol-mBC \times Dsol-nBC==0;
solutionD=Solve[{eqnD1,eqnD2},{xDsol,yDsol}];
\]

where $mBC$ is the slope and $bBC$ is the $y$-intercept of the line $BC$.

Two sets of solutions are found for the position of the joint $D$, that are functions of the angle $\phi(t)$, i.e. functions of time.

\[
xD1=xDsol/.solutionD[[1]];
yD1=yDsol/.solutionD[[1]];
xD2=xDsol/.solutionD[[2]];
yD2=yDsol/.solutionD[[2]];
\]

To determine the correct position of the joint $D$ for the mechanism, an additional condition is needed. For the first quadrant, $0 \leq \phi \leq 90^\circ$, the condition is $x_D \leq x_C$.

This condition using the Mathematica™ command is

\[
\text{If[xD1/.initdata<=xC, xD=xD1;yD=yD1, xD=xD2;yD=yD2]};
\]
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The numerical solutions are printed using Mathematica™

\[
\text{Print["xD = ", xD/.initdata, " m"]};
\text{Print["yD = ", yD/.initdata, " m"]};
\]

The linear velocity vector of the joint \( D \) (\( D = D_3 = D_4 \) on link 3 or link 4) is

\[
v_D = v_{D_3} = v_{D_4} = \dot{x}_D \hat{i} + \dot{y}_D \hat{j},
\]

where

\[
\dot{x}_D = \frac{dx_D}{dt} \quad \text{and} \quad \dot{y}_D = \frac{dy_D}{dt}.
\]

are the components of the velocity vector of the joint \( D \) on the \( x \)-axis and, respectively, on the \( y \)-axis.

To calculate symbolically the components of this velocity vector the following Mathematica™ commands are used

\[
\begin{align*}
\text{vDx} &= \text{D[xD,t]}; \\
\text{vDy} &= \text{D[yD,t]};
\end{align*}
\]

For the considered mechanism the numerical values are

\[
\dot{x}_D = 0.067 \text{ m/s} \quad \text{and} \quad \dot{y}_D = -0.814 \text{ m/s}.
\]

The numerical solutions are printed using Mathematica™

\[
\text{Print["vDx = ", vDx/.initdata, " m/s"]};
\text{Print["vDy = ", vDy/.initdata, " m/s"]};
\]

The linear acceleration vector of \( D = D_3 = D_4 \) is

\[
a_D = \ddot{x}_D \hat{i} + \ddot{y}_D \hat{j},
\]

where

\[
\ddot{x}_D = \frac{d\dot{x}_D}{dt} \quad \text{and} \quad \ddot{y}_B = \frac{d\dot{y}_B}{dt}.
\]

To calculate symbolically the components of the acceleration vector the following Mathematica™ commands are used

\[
\text{aDx} = \text{D[vDx,t]};
\]
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\[ aDy = D[vDy, t]; \]

The numerical values of the acceleration of \( D \) are

\[ \ddot{x}_D = 4.617 \text{ m/s}^2 \quad \text{and} \quad \ddot{y}_D = -1.811 \text{ m/s}^2, \]

and can be printed using Mathematica\textsuperscript{TM}:

\begin{verbatim}
Print["aDx = ", aDx/.initdata, " m/s^2 ";
Print["vDy = ", vDy/.initdata, " m/s^2 ";
\end{verbatim}

The angle \( \phi_3(t) \) is determined as a function of time \( t \) from the equation of the slope of the line \( BC \)

\[ \tan \phi_3(t) = m_{BC}(t). \]

The Mathematica\textsuperscript{TM} function \texttt{ArcTan[z]}, gives the arc tangent of the number \( z \). To calculate symbolically the angle \( \phi_3 \)

\[ \texttt{phi3=ArcTan[mBC]}; \]

The angular velocity \( \omega_3(t) \) is the derivative with respect to time of the angle \( \phi_3(t) \)

\[ \omega_3 = \frac{d\phi_3(t)}{dt}. \]

Symbolically the angular velocity \( \omega_3 \) is calculated using Mathematica\textsuperscript{TM}:

\[ \texttt{omega3=D[phi3,t]}; \]

The angular acceleration \( \alpha_3(t) \) is the derivative with respect to time of the angular velocity \( \omega_3(t) \)

\[ \alpha_3(t) = \frac{d\omega_3(t)}{dt}. \]

Symbolically, using Mathematica\textsuperscript{TM}, the angular acceleration \( \alpha_3 \) is

\[ \texttt{alpha3=D[omega3,t]}; \]
The numerical values of the angles, angular velocities and angular accelerations for the link 2 and link 3 are

\[ \phi_3 = \phi_2 = 0.082 \text{ rad}, \quad \omega_3 = \omega_2 = 5.448 \text{ rad/s}, \quad \alpha_3 = \alpha_2 = 14.568 \text{ rad/s}^2. \]

The numerical solutions are printed using Mathematica\textsuperscript{TM}

\[
\text{Print["phi3=phi2= ", phi3/.initdata," rad "];}
\text{Print["omega3=omega2= ", omega3/.initdata," rad/s"]};
\text{Print["alpha3=alpha2= ", alpha3/.initdata," rad/s}^2"]};
\]

The angle \( \phi_5(t) \) is determined as a function of time \( t \) from the following equation

\[ \tan \phi_5(t) = \frac{y_D(t) - y_E}{x_D(t) - x_E}, \]

and symbolically using Mathematica\textsuperscript{TM}

\[
\phi_5 = \text{ArcTan}\left(\frac{y_D-y_E}{x_D-x_E}\right);
\]

The angular velocity \( \omega_5(t) \) is the derivative with respect to time of the angle \( \phi_5(t) \)

\[ \omega_5 = \frac{d\phi_5(t)}{dt}. \]

To calculate symbolically the angular velocity \( \omega_5 \) using Mathematica\textsuperscript{TM} the following command is used

\[
\omega_5 = \text{D[phi5,t]};
\]

The angular acceleration \( \alpha_5(t) \) is the derivative with respect to time of the angular velocity \( \omega_5(t) \)

\[ \alpha_5(t) = \frac{d\omega_5(t)}{dt}, \]

and symbolically with Mathematica\textsuperscript{TM}

\[
\alpha_5 = \text{D[omega5,t]};
\]
The numerical values of the angles, angular velocities and angular accelerations for the link 5 and link 4 are

\[ \phi_5 = \phi_4 = 2.036 \text{ rad}, \quad \omega_5 = \omega_4 = 0.917 \text{ rad/s}, \quad \alpha_5 = \alpha_4 = -5.771 \text{ rad/s}^2. \]

The numerical solutions printed with Mathematica™ are

```
Print["phi5=phi4= ",phi5/.initdata," rad "];
Print["omega5=omega4= ",omega5/.initdata," rad/s"];
Print["alpha5=alpha4= ",alpha5/.initdata," rad/s^2"];
```

The Mathematica™ program for the velocity and acceleration analysis is given in Program 7.4.

### 7.3 Contour Equations for Velocities and Accelerations

The same planar R-RTR-RTR mechanism is considered in Fig. 7.8(a). The driver link 1 is rotating with a constant speed of \( n = 50 \text{ rpm} \). A Mathematica™ program for velocity and acceleration analysis using the contour equations will be presented.

The mechanism has five moving links and seven full joints. The number of independent contours is

\[ n_c = c - n = 7 - 5 = 2, \]

where \( c \) is the number of joints and \( n \) is the number of moving links.

The mechanism has two independent contours. The first contour \( I \) contains the links 0, 1, 2 and 3, while the second contour \( II \) contains the links 0, 3, 4 and 5. The diagram of the mechanism is represented in Fig. 7.8(b). Clockwise paths are chosen for each closed contours \( I \) and \( II \).

**First contour analysis**

Figure 7.9(a) shows the first independent contour \( I \) with

- rotational joint R between the links 0 and 1 (joint \( A \));
- rotational joint R between the links 1 and 2 (joint \( B \));
- translational joint T between the links 2 and 3 (joint \( B \));
- rotational joint R between the links 3 and 0 (joint \( C \)).

The angular velocity \( \omega_{10} \) of the driver link is known

\[ \omega_{10} = \omega_1 = \omega = \frac{50\pi}{30} \text{ rad/s} = 5.235 \text{ rad/s}. \]
The origin of the reference frame is the point \( A(0,0,0) \).

For the velocity analysis, the following vectorial equations are used

\[
\begin{align*}
\omega_{10} + \omega_{21} + \omega_{03} &= 0, \\
r_{AB} \times \omega_{21} + r_{AC} \times \omega_{03} + v_{B32}^r &= 0,
\end{align*}
\]

where \( r_{AB} = x_B \mathbf{i} + y_B \mathbf{j} \), \( r_{AC} = x_C \mathbf{i} + y_C \mathbf{j} \), and

\[
\begin{align*}
\omega_{10} &= \omega_{10} \mathbf{k}, \quad \omega_{21} = \omega_{21} \mathbf{k}, \quad \omega_{03} = \omega_{03} \mathbf{k}, \\
v_{B32}^r = v_{32} \cos \phi_2 \mathbf{i} + v_{32} \sin \phi_2 \mathbf{j}.
\end{align*}
\]

The sign of the relative angular velocities is selected as positive as shown in Figs. 7.8(a) and 7.9(a). Then the numerical computation will give the correct orientation of the unknown vectors. The components of the vectors \( r_{AB} \) and \( r_{AC} \), and the angle \( \phi_2 \) are already known from the position analysis of the mechanism. Equation (7.9) becomes

\[
\omega_{10} \mathbf{k} + \omega_{21} \mathbf{k} + \omega_{03} \mathbf{k} = 0,
\]

In the Mathematica\textsuperscript{TM} environment, a three-dimensional vector \( \mathbf{v} \) is written as a list of variables \( \mathbf{v} = \{x, y, z\} \) where \( x, y, \) and \( z \) are the spatial coordinates of the vector \( \mathbf{v} \). The first component of the vector \( \mathbf{v} \) is \( x = \mathbf{v}[1] \), the second component is \( y = \mathbf{v}[2] \), and the third component is \( z = \mathbf{v}[3] \). For the considered mechanism with Mathematica\textsuperscript{TM}

\[
\begin{align*}
\text{rB} &= \{x_B, y_B, 0\}; \\
\text{rC} &= \{x_C, y_C, 0\}; \\
\text{omega10v} &= \{0, 0, \omega_{10}\}; \\
\text{omega21vSol} &= \{0, 0, \omega_{21}\}; \\
\text{omega03vSol} &= \{0, 0, \omega_{03}\}; \\
\text{v32vSol} &= \{v_{32} \cos \phi_2, v_{32} \sin \phi_2, 0\};
\end{align*}
\]

Equation (7.10) represents a system of three equations and with Mathematica\textsuperscript{TM} commands gives

\[
\text{eqIkv} = (\text{omega10v} + \text{omega21vSol} + \text{omega03vSol})[[3]] == 0;
\]
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\[
eq I_{Iv} = (\text{Cross}[r_B, \omega_{21vSol}] + \text{Cross}[r_C, \omega_{03vSol}] + v_{32vSol})[[1]] = 0;
\]

\[
eq I_{Ijv} = (\text{Cross}[r_B, \omega_{21vSol}] + \text{Cross}[r_C, \omega_{03vSol}] + v_{32vSol})[[2]] = 0;
\]

where the command \text{Cross}[a,b] gives the vector cross product of the vectors \(a\) and \(b\).

The system of equations can be solved using the Mathematica™ commands

\[
\text{solIvel = Solve}\left\{\{\text{eqIkv}, \text{eqIiv}, \text{eqIjv}\}, \{\omega_{21Sol}, \omega_{03Sol}, v_{32Sol}\}\right\};
\]

The following numerical solutions are obtained

\[
\omega_{21} = 0.212 \text{ rad/s}, \ \omega_{03} = -5.448 \text{ rad/s}, \ \text{and} \ v_{32} = 0.313 \text{ m/s}.
\]

To print the numerical values the following Mathematica™ commands are used

\[
\text{omega21v = omega21vSol/.solIvel[[1]]};
\]
\[
\text{omega03v = omega03vSol/.solIvel[[1]]};
\]
\[
v_{32v} = v_{32vSol/.solIvel[[1]]};
\]
\[
\text{Print["\omega_{21} = ",omega21v];}
\]
\[
\text{Print["\omega_{03} = ",omega03v];}
\]
\[
\text{Print["v_{32} = ",v_{32v];}
\]
\[
\text{Print["v_{32r} = ",v_{32Sol/.solIvel[[1]]}];}
\]

The absolute angular velocities of the links 2 and 3 are

\[
\omega_{20} = \omega_{30} = -\omega_{03} = 5.448 \text{ k rad/s}.
\]

The absolute linear velocities of the joints \(B\) and \(D\) are

\[
v_B = v_{B1} = v_{B2} = v_A + \omega_{10} \times r_{AB} = -0.366 \text{ i} + 0.634 \text{ j m/s},
\]
\[
v_D = v_{D3} = v_{D4} = v_C + \omega_{30} \times r_{CD} = 0.067 \text{ i} - 0.814 \text{ j m/s},
\]

where \(v_A = 0\) and \(v_C = 0\) because the joints \(A\) and \(C\) are grounded and \(r_{CD} = r_{AD} - r_{AC}\).
The MathematicaTM commands for the absolute velocities are

\begin{verbatim}
omega20v=omega30v=-omega03v;
vBv=Cross[omega10v,rB];
vDv=Cross[omega30v,(rD-rC)];
Print["omega20 = omega30 = ",omega20v];
Print["vB = ",vBv];
Print["vD = ",vDv];
\end{verbatim}

For the acceleration analysis, the following vectorial equations are used

\[ \alpha_{10} + \alpha_{21} + \alpha_{03} = 0, \]
\[ r_{AB} \times \alpha_{21} + r_{AC} \times \alpha_{03} + a_{B32}^r + a_{B32}^c - \omega_{10}^2 r_{AB} - \omega_{20}^2 r_{BC} = 0. \] (7.11)

where

\[ \alpha_{10} = \alpha_{10} k, \quad \alpha_{21} = \alpha_{21} k, \quad \alpha_{03} = \alpha_{03} k, \]
\[ a_{B32}^r = a_{32} = a_{32} \cos \phi_2 i + a_{32} \sin \phi_2 j, \]
\[ a_{B32}^c = a_{32}^c = 2 \omega_{20} \times v_{32}, \]

The driver link has a constant angular velocity and \( \alpha_{10} = \dot{\omega}_{10} = 0. \)

The acceleration vectors using the MathematicaTM commands are

\begin{verbatim}
alpha10v={0,0,0};
alpha21vSol={0,0,alpha21Sol};
alpha03vSol={0,0,alpha03Sol};
a32vSol={a32Sol Cos[phi2],a32Sol Sin[phi2],0};
\end{verbatim}

Equation (7.11) represents a system of three equations and using MathematicaTM commands gives

\begin{verbatim}
eqIka=(alpha10v+alpha21vSol+alpha03vSol)[[3]]==0;
eqIia=( Cross[rB,alpha21vSol]+Cross[rC,alpha03vSol]+a32vSol+2Cross[omega20v,v32v]-(omega10v.omega10v)rB-(omega20v.omega20v)(rC-rB) )[[1]]==0;
eqIja=( Cross[rB,alpha21vSol]+Cross[rC,alpha03vSol]+a32vSol+2Cross[omega20v,v32v]-(omega10v.omega10v)rB-
\end{verbatim}
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\[(\omega_20v . \omega_20v) (rC-rB) \] [[2]] == 0;

The unknowns in the Eq. (7.18) are $\alpha_{21}$, $\alpha_{03}$ and $a_{32}$. The system of equations is solved using the Mathematica\textsuperscript{TM} commands

\[
solIacc=Solve[\{eqIka, eqIia, eqIja\}, \{alpha21Sol, alpha03Sol, a32Sol\}];
\]

The following numerical solutions are obtained

$\alpha_{21} = 14.568 \text{ rad/s}^2$, $\alpha_{03} = -14.568 \text{ rad/s}^2$, and $a_{32} = -0.140 \text{ m/s}^2$.

To print the numerical values the following Mathematica\textsuperscript{TM} commands are used

\[
alpha21v=alpha21vSol/.solIacc[[1]]; \\
alpha03v=alpha03vSol/.solIacc[[1]]; \\
a32v=a32vSol/.solIacc[[1]]; \\
Print["alpha21 = ",alpha21v]; \\
Print["alpha03 = ",alpha03v]; \\
Print["a32 = ",a32v]; \\
Print["a32r = ",a32Sol/.solIacc[[1]]];
\]

The absolute angular accelerations of the links 2 and 3 are

$\alpha_{20} = \alpha_{30} = -\alpha_{03} = 14.568 \text{ k rad/s}^2$.

The absolute linear accelerations of the joints $B$ and $D$ are obtained from the following equation

\[
a_B = a_A + \alpha_{10} \times r_{AB} - \omega_{10}^2 AB = -3.323 \mathbf{i} - 1.919 \text{ J m/s}^2, \\
a_D = a_C + \alpha_{30} \times r_{CD} - \omega_{30}^2 r_{CD} = 4.617 \mathbf{i} - 1.811 \text{ J m/s}^2,
\]

where $a_A = 0$ and $a_C = 0$ because the joints $A$ and $C$ are grounded.

To print the absolute accelerations with Mathematica\textsuperscript{TM} the following relations are used

\[
alpha20v=alpha30v=-alpha03v; \\
aBv=-(omega10v . omega10v)rB;
\]
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\[ aDv = \text{Cross}[\alpha_{30v}, (rD-rC)] - \omega_{20v}.\omega_{20v} (rD-rC); \]
\[ \text{Print["\alpha_{20} = \alpha_{30} = ",}\alpha_{30v}]; \]
\[ \text{Print["aB = ",}aBv]; \]
\[ \text{Print["aD = ",}aDv]; \]

Second contour analysis

Figure 7.10(a) depicts the second independent contour

- rotational joint R between the links 0 and 3 (joint C);
- rotational joint R between the links 3 and 4 (joint D);
- translational joint T between the links 4 and 5 (joint D);
- rotational joint R between the links 5 and 0 (joint E).

For the velocity analysis, the following vectorial equations are used

\[ \omega_{30} + \omega_{43} + \omega_{05} = 0, \]
\[ r_{AC} \times \omega_{30} + r_{AD} \times \omega_{43} + r_{AE} \times \omega_{05} + v_{rD}^r = 0, \]

where \( r_{AD} = x_D \hat{i} + y_D \hat{j}, \)
\( r_{AE} = x_E \hat{i} + y_E \hat{j}, \)
\( \omega_{30} = \omega_{30} \hat{k}, \)
\( \omega_{43} = \omega_{43} \hat{k}, \)
\( \omega_{05} = \omega_{05} \hat{k}, \)
\( v_{rD}^r = v_{54} \cos \phi_4 \hat{i} + v_{54} \sin \phi_4 \hat{j}. \)

The sign of the relative angular velocities is selected as positive as shown in Figs. 7.8(a) and 7.10(a). Then the numerical computation will give the correct orientation of the unknown vectors. The components of the vectors \( r_{AD} \) and \( r_{AE}, \) and the angle \( \phi_4 \) are already known from the position analysis of the mechanism.

The unknown vectors with Mathematica\textsuperscript{TM} commands are

\[ \text{omega43vSol=\{0,0,omega43Sol\};} \]
\[ \text{omega05vSol=\{0,0,omega05Sol\};} \]
\[ \text{v54vSol=\{v54Sol Cos[phi4],v54Sol Sin[phi4],0\};} \]

Equations (7.12) becomes

\[ \omega_{30} \hat{k} + \omega_{43} \hat{k} + \omega_{05} \hat{k} = 0, \]
\[ \begin{pmatrix} 1 & 1 & 1 \\ x_C & y_C & 0 \\ 0 & 0 & \omega_{30} \end{pmatrix} + \begin{pmatrix} 1 & 1 & 1 \\ x_D & y_D & 0 \\ 0 & 0 & \omega_{43} \end{pmatrix} + \begin{pmatrix} 1 & 1 & 1 \\ x_E & y_E & 0 \\ 0 & 0 & \omega_{05} \end{pmatrix} + \]
\[ v_{32} \cos \phi_4 \hat{i} + v_{32} \sin \phi_4 \hat{j} = 0. \]
The Eq. (7.13) projected onto the “fixed” reference frame $Oxyz$ gives

\[
\omega_{30} + \omega_{43} + \omega_{05} = 0,
\]

\[
y_C\omega_{30} + y_D\omega_{43} + y_E\omega_{05} + v_{54}\cos \phi_4 = 0,
\]

\[
-x_C\omega_{30} - x_D\omega_{43} - x_E\omega_{05} + v_{54}\sin \phi_4 = 0.
\]

(7.14)

The above system of equations using the following Mathematica\textsuperscript{TM} commands becomes

\[
eq II_{kv} = (\omega_{30}v + \omega_{43}v_{Sol} + \omega_{05}v_{Sol})[[3]] = 0;
\]

\[
eq II_{iv} = (\text{Cross}[rC, \omega_{30}v] + \text{Cross}[rD, \omega_{43}v_{Sol}] + \\
\text{Cross}[rE, \omega_{05}v_{Sol}] + v_{54}v_{Sol})[[1]] = 0;
\]

\[
eq II_{jv} = (\text{Cross}[rC, \omega_{30}v] + \text{Cross}[rD, \omega_{43}v_{Sol}] + \\
\text{Cross}[rE, \omega_{05}v_{Sol}] + v_{54}v_{Sol})[[2]] = 0;
\]

The Eq. (7.14) represents an algebraic system of three equations with three unknowns $\omega_{43}$, $\omega_{05}$ and $v_{54}$. The system is solved using the Mathematica\textsuperscript{TM} commands

\[
solIIvel = \text{Solve}\{\{\text{eqII}_{kv}, \text{eqII}_{iv}, \text{eqII}_{jv}\}, \{\omega_{43}v_{Sol}, \\
\omega_{05}v_{Sol}, v_{54}v_{Sol}\}\};
\]

The following numerical solutions are obtained

\[
\omega_{43} = -4.531 \text{ rad/s}, \quad \omega_{05} = -0.917 \text{ rad/s}, \quad \text{and} \quad v_{54} = 0.757 \text{ m/s}.
\]

To print the numerical values with Mathematica\textsuperscript{TM} the following commands are used

\[
\omega_{43}v = \omega_{43}v_{Sol}/.\text{solIIvel}[[1]];
\]

\[
\omega_{05}v = \omega_{05}v_{Sol}/.\text{solIIvel}[[1]];
\]

\[
v_{54}v = v_{54}v_{Sol}/.\text{solIIvel}[[1]];
\]

\[
\text{Print}[^{\omega_{43} = }", \omega_{43}v];
\]

\[
\text{Print}[^{\omega_{05} = }, \omega_{05}v];
\]

\[
\text{Print}[^{v_{54} = }", v_{54}v];
\]

\[
\text{Print}[^{v_{54}r = }", v_{54}v_{Sol}/.\text{solIIvel}[[1]] ];
\]

The absolute angular velocities of the links 4 and 5 are

\[
\omega_{40} = \omega_{50} = -\omega_{05} = 0.917 \text{ k rad/s},
\]

(7.15)
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and with Mathematica™ commands

\[
\begin{align*}
\omega_{40}v &= \omega_{50}v = -\omega_{05}v; \\
\text{Print}["\omega_{40} = \omega_{50} = ", \omega_{50}v];
\end{align*}
\]

For the acceleration analysis, the following vectorial equations are used

\[
\alpha_{30} + \alpha_{43} + \alpha_{05} = 0, \tag{7.16}
\]

\[
r_{AC} \times \alpha_{30} + r_{AD} \times \alpha_{43} + r_{AE} \times \alpha_{05} + a_{D54}^r + a_{cB54}^r - \omega_{30}^2 r_{CD} - \omega_{40}^2 r_{DE} = 0.
\]

where

\[
\begin{align*}
\alpha_{30} &= \alpha_{30}k, \\
\alpha_{43} &= \alpha_{43}k, \\
\alpha_{05} &= \alpha_{05}k, \\
a_{B54}^r &= a_{54} = a_{54} \cos \phi_4 \mathbf{i} + a_{54} \sin \phi_4 \mathbf{j}, \\
a_{cB54}^c &= 2\omega_{40} \times v_{54}.
\end{align*}
\]

The unknown acceleration vectors using the Mathematica™ commands are

\[
\begin{align*}
\alpha_{43}vSol &= \{0, 0, \alpha_{43}Sol\}; \\
\alpha_{05}vSol &= \{0, 0, \alpha_{05}Sol\}; \\
a_{54}vSol &= \{a_{54}Sol \ \cos[\phi_4], a_{54}Sol \ \sin[\phi_4], 0\};
\end{align*}
\]

Equation (7.16) becomes

\[
\begin{align*}
\alpha_{30}k + \alpha_{43}k + \alpha_{05}k &= 0, \\
\begin{vmatrix}
1 & 1 & 1 \\
x_C & y_C & x_D \\
0 & 0 & \alpha_{30}
\end{vmatrix}
+ \begin{vmatrix}
1 & 1 & 1 \\
x_D & y_D & x_E \\
0 & 0 & \alpha_{43}
\end{vmatrix}
+ \begin{vmatrix}
1 & 1 & 1 \\
x_E & y_E & y_{05} \\
0 & 0 & \alpha_{05}
\end{vmatrix}
+ \\
\begin{align*}
&\begin{vmatrix}
1 & 1 & 1 \\
\alpha_{30} & \alpha_{43} & \alpha_{05}
\end{vmatrix}
+ \\
&\begin{vmatrix}
1 & 1 & 1 \\
a_{cB54} & v_{54} \cos \phi_4 & v_{54} \sin \phi_4 \\
v_{54} \cos \phi_4 & v_{54} \sin \phi_4 & 0
\end{vmatrix}
+ \\
&\begin{vmatrix}
1 & 1 & 1 \\
\omega_{30}^2 & \omega_{40}^2 \\
(x_D - x_C) & (y_D - y_C)
\end{vmatrix}
+ \\
&\begin{vmatrix}
1 & 1 & 1 \\
\omega_{40}^2 & \omega_{40}^2 \\
(x_E - x_D) & (y_E - y_D)
\end{vmatrix} = 0.
\end{align*}
\]

\]

The Eq. (7.17) can be rewritten as

\[
\begin{align*}
\alpha_{30} + \alpha_{43} + \alpha_{05} &= 0, \\
y_C \alpha_{30} + y_D \alpha_{43} + y_E \alpha_{05} + a_{54} \cos \phi_4 - 2\omega_{40}v_{54} \sin \phi_4 &= 0.
\end{align*}
\]
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\[
\begin{align*}
\omega^2_{30}(x_D - x_C) - \omega^2_{40}(x_E - x_D) &= 0, \\
-x_C\alpha_{30} - x_D\alpha_{43} - x_E\alpha_{05} + a_{54}\sin\phi_4 + 2\omega_{40}v_{54}\cos\phi_4 - \\
\omega^2_{30}(y_D - y_C) - \omega^2_{40}(y_E - y_D) &= 0. \\
\end{align*}
\] (7.18)

The contour acceleration equations using Mathematica™ commands are

\[
\begin{align*}
\text{eqIIka} &= (\alpha_{30}v + \alpha_{43}v_{Sol} + \alpha_{05}v_{Sol})[[3]] == 0; \\
\text{eqIIia} &= \left( Cross[rC, \alpha_{30}v] + Cross[rD, \alpha_{43}v_{Sol}] + Cross[rE, \alpha_{05}v_{Sol}] + a_{54}v_{Sol} + 2Cross[\omega_{40}v, v_{54}v] - (\omega_{30}v.\omega_{30}v)(rD-rC) - (\omega_{40}v.\omega_{40}v)(rE-rD) \right) [[1]] == 0; \\
\text{eqIIja} &= \left( Cross[rC, \alpha_{30}v] + Cross[rD, \alpha_{43}v_{Sol}] + Cross[rE, \alpha_{05}v_{Sol}] + a_{54}v_{Sol} + 2Cross[\omega_{40}v, v_{54}v] - (\omega_{30}v.\omega_{30}v)(rD-rC) - (\omega_{40}v.\omega_{40}v)(rE-rD) \right) [[2]] == 0;
\end{align*}
\]

The unknowns in the Eq. (7.18) are \(\alpha_{43}, \alpha_{05}\) and \(a_{54}\). To solve the system the following Mathematica™ command is used

\[
\text{solIIacc} = \text{Solve}\{\text{eqIIka, eqIIia, eqIIja}, \{\alpha_{43}Sol, \alpha_{05}Sol, a_{54}Sol\}\};
\]

The following numerical solutions are obtained

\[
\begin{align*}
\alpha_{43} &= -20.339 \text{ rad/s}^2, \quad \alpha_{05} = 5.771 \text{ rad/s}^2, \quad \text{and} \quad a_{54} = 3.411 \text{ m/s}^2.
\end{align*}
\]

The Mathematica™ commands are

\[
\begin{align*}
\alpha_{43}v &= \alpha_{43}v_{Sol} /. \text{solIIacc}[[1]]; \\
\alpha_{05}v &= \alpha_{05}v_{Sol} /. \text{solIIacc}[[1]]; \\
a_{54}v &= a_{54}v_{Sol} /. \text{solIIacc}[[1]]; \\
\text{Print}["\alpha_{43} = ", \alpha_{43}v]; \\
\text{Print}["\alpha_{05} = ", \alpha_{05}v]; \\
\text{Print}["a_{54} = ", a_{54}v]; \\
\text{Print}["a_{54}r = ", a_{54}Sol /. \text{solIIacc}[[1]]];
\end{align*}
\]
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The absolute angular accelerations of the links 4 and 5 are
\[
\alpha_{40} = \alpha_{50} = -\alpha_{05} = -5.771 \text{ k rad/s}^2,
\]
and with Mathematica™

\[
\text{alpha40v} = \text{alpha50v} = -\text{alpha05v};
\]
\[
\text{Print} ["\alpha_{40} = \alpha_{50} = \text{, alpha50v}];
\]

The Mathematica™ program for the velocity and acceleration analysis using the contour method is given in Program 7.5.

7.4 Dynamic Force Analysis

In this section, the motor moment \( M_m \) required for the dynamic equilibrium of the considered mechanism, shown in Fig. 7.11(a), is calculated. The joint reaction forces are also calculated. The widths of the links 1, 3, and 5 are \( AB = 0.140 \text{ m}, FD = 0.400 \text{ m} \), and respectively, \( EG = 0.500 \text{ m} \). The height of the links 1, 3, and 5 is \( h = 0.010 \text{ m} \). The width of the links 2 and 4 is \( w_{\text{Slider}} = 0.050 \text{ m} \), and the height is \( h_{\text{Slider}} = 0.020 \text{ m} \). All the five moving links are rectangular prisms with the depth \( d = 0.001 \text{ m} \). The angle of the driver link is \( \phi = \frac{\pi}{6} \text{ rad} \) and the angular velocity is \( n = 50 \text{ rpm} \). The external moment applied on the link 5 is opposed to the motion of the link. Because \( \omega_5 = 0.917 \text{ k rad/s} \), the external moment vector will be \( \text{Me} = -100 \text{ k N·m} \). The density of the material is \( \rho_{\text{Steel}} = \rho = 8000 \text{ kg/m}^3 \). The gravitational acceleration is \( g = 9.807 \text{ m/s}^2 \). The center of mass locations of the links \( i = 1, 2, ..., 5 \) are designated by \( C_i(x_{C_i}, y_{C_i}, 0) \).

The input data are introduced using a Mathematica™ rule

\[
\text{rule} = \{\text{AB} \to 0.14, \text{AC} \to 0.06, \text{AE} \to 0.25, \text{CD} \to 0.15, \text{FD} \to 0.4, \text{EG} \to 0.5, h \to 0.01, d \to 0.001, h_{\text{Slider}} \to 0.02, w_{\text{Slider}} \to 0.05, \rho_o \to 8000, g \to 9.807, Me \to -100.\}
\]
\[
\text{phi[t]} \to \text{N}[\text{Pi}]/6, \text{phi’[t]} \to \text{omega}, \text{phi’’[t]} \to 0\};
\]

where \( \text{omega} = n \times \text{N}[\text{Pi}]/30 \).

Inertia forces and moments
To calculate the inertia moment $M_i$ and the total force $F_i$ for the link $i = 1, 2, ..., 5$, the mass $m_i$, the acceleration of the center of mass $a_{C_i}$, the gravity force $G_i$, and the mass moment of inertia $I_{C_i}$ are needed.

**Link 1**

The mass of the link is

$$m_1 = \rho \cdot AB \cdot h \cdot d.$$ 

The position, velocity, and acceleration for the center of mass $C_1$ are

$$r_{C_1} = r_B/2, \quad v_{C_1} = v_B/2, \quad \text{and} \quad a_{C_1} = a_B/2.$$ 

The inertia force is

$$F_{in1} = -m_1 \cdot a_{C_1}.$$ 

The gravitational force is

$$G_1 = -m_1 \cdot g \cdot k.$$ 

The total force on link 1 at the mass center $C_1$ is

$$F_1 = F_{in1} + G_1.$$ 

The mass moment of inertia is

$$I_{C_1} = m_1 \cdot (AB^2 + h^2)/12.$$ 

The moment of inertia is

$$M_1 = M_{in1} = -I_{C_1} \cdot \alpha_1.$$ 

To calculate and print the numerical values of the total force $F_1$ and the moment $M_1$, the following *Mathematica* commands are used

```mathematica
ml=rho AB h d /. rule;
rCl=rB/2; vCl=vB/2; aCl=aB/2;
Fin1=-ml aCl /. rule;
G1={0,-ml*g,0} /. rule;
F1=(Fin1+G1) /. rule;
IC1=ml (AB^2+h^2)/12 /. rule;
M1=Min1=-IC1 alpha1 /. rule;
Print["F1 = ",F1];
Print["M1 = ",M1];
```
Link 2
The mass of the link 2 is

\[ m_2 = \rho \ h_{\text{slider}} \ w_{\text{slider}} \ d. \]

The position, velocity, and acceleration for the center of mass \( C_2 \) are

\[ r_{C_2} = r_B, \ v_{C_2} = v_B, \ \text{and} \ a_{C_2} = a_B. \]

The inertia force is

\[ F_{in2} = -m_2 \ a_{C_2}. \]

The gravitational force is

\[ G_2 = -m_2 \ g \ k. \]

The total force on slider 2 at \( B \) is

\[ F_2 = F_{in2} + G_2. \]

The mass moment of inertia is

\[ I_{C_2} = m_2 \ (h_{\text{slider}}^2 + w_{\text{slider}}^2)/12. \]

The moment of inertia is

\[ M_2 = M_{in2} = -I_{C_2} \alpha_2. \]

The Mathematica\textsuperscript{TM} commands for the total force \( F_2 \) and the moment \( M_2 \) are

\[
\begin{align*}
m2 &= \text{rho} \ h_{\text{slider}} \ w_{\text{slider}} \ d /. \text{rule}; \\
rC2 &= rB; \ vC2 = vB; \ aC2 = aB; \\
Fin2 &= -m2 \ aC2 /. \text{rule}; \\
G2 &= \{0, -m2*g, 0\} /. \text{rule}; \\
F2 &= (\text{Fin2} + G2) /. \text{rule}; \\
IC2 &= m2 \ (h_{\text{slider}}^2 + w_{\text{slider}}^2)/12 /. \text{rule}; \\
M2 &= \text{Min2} = -IC2 \ \text{alpha2} /. \text{rule}; \\
\text{Print}['F2 = ', F2]; \\
\text{Print}['M2 = ', M2];
\end{align*}
\]

Link 3
The mass of link 3 is

\[ m_3 = \rho \ FD \ h \ d. \]
The position, velocity, and acceleration for the center of mass $C_3$ are

\begin{align*}
x_{C_3} &= x_C + (FD/2 - CD) \cos \phi_3, \quad y_{C_3} = y_C + (FD/2 - CD) \sin \phi_3, \\
r_{C_3} &= x_{C_3} \mathbf{i} + y_{C_3} \mathbf{j}, \quad \mathbf{v}_{C_3} = \dot{x}_{C_3} \mathbf{i} + \dot{y}_{C_3} \mathbf{j}, \quad \mathbf{a}_{C_3} = \ddot{x}_{C_3} \mathbf{i} + \ddot{y}_{C_3} \mathbf{j}.
\end{align*}

The inertia force is \( \mathbf{F}_{in3} = -m_3 \mathbf{a}_{C_3}. \)

The gravitational force is \( \mathbf{G}_3 = -m_3 g \mathbf{k}. \)

The total force at \( C_3 \) is \( \mathbf{F}_3 = \mathbf{F}_{in3} + \mathbf{G}_3. \)

The mass moment of inertia is

\[ I_{C_3} = m_3 (FD^2 + h^2)/12. \]

The total moment on link 3 is \( M_3 = M_{in3} = -I_{C_3} \alpha_3. \)

The force \( \mathbf{F}_3 \) and the moment \( M_3 \) with Mathematica\textsuperscript{TM} are

\begin{verbatim}
m3=rho FD h d /.rule; xC3=xC+(FD/2-CD) Cos[phi3]; yC3=yC+(FD/2-CD) Sin[phi3]; rC3={xC3,yC3,0}; vC3=D[rC3,t]; aC3=D[D[rC3,t],t]; Fin3=-m3 aC3 /.rule; G3={0,-m3*g,0} /.rule; F3=(Fin3+G3) /.rule; IC3=m3 (FD^2+h^2)/12 /.rule; M3=Min3=-IC3 alpha3 /.rule; Print["F3 = ",F3]; Print["M3 = ",M3];
\end{verbatim}

**Link 4**

The mass of link 4 is \( m_4 = \rho \, h_{Slider} \, w_{Slider} \, d. \)
The position, velocity, and acceleration for the center of mass \( C_4 \) are

\[
\mathbf{r}_{C_4} = \mathbf{r}_D, \quad \mathbf{v}_{C_4} = \mathbf{v}_D, \quad \text{and} \quad \mathbf{a}_{C_4} = \mathbf{a}_D.
\]

The inertia force is

\[
\mathbf{F}_{in4} = -m_4 \mathbf{a}_{C_4}.
\]

The gravitational force is

\[
\mathbf{G}_4 = -m_4 \mathbf{g} \mathbf{k}.
\]

The total force on slider 4 at \( D \) is

\[
\mathbf{F}_4 = \mathbf{F}_{in4} + \mathbf{G}_4.
\]

The mass moment of inertia is

\[
I_{C_4} = m_4 \left( h_{Slider}^2 + w_{Slider}^2 \right)/12.
\]

The moment of inertia is

\[
M_4 = M_{in4} = -I_{C_4} \alpha_4.
\]

To calculate and print the numerical values of the total force \( \mathbf{F}_4 \) and the moment \( M_4 \) the following Mathematica\textsuperscript{TM} commands are used

\[
\begin{align*}
m4 &= \rho \text{ hSlider wSlider d} /. \text{rule}; \\
rC4 &= \text{rD}; \quad vC4 = \text{vD}; \quad aC4 = \text{aD}; \\
Fin4 &= -m4 \text{ aC4} /. \text{rule}; \\
G4 &= \{0, -m4 * \text{g, 0} \} /. \text{rule}; \\
F4 &= (\text{Fin4+G4}) /. \text{rule}; \\
IC4 &= m4 \left( h_{Slider}^2 + w_{Slider}^2 \right)/12 /. \text{rule}; \\
M4 &= \text{Min4} = -IC4 \text{ alpha4} /. \text{rule}; \\
\text{Print["F4 = ",F4];} \\
\text{Print["M4 = ",M4];}
\end{align*}
\]

\textbf{Link 5}

The mass of link 5 is

\[
m_5 = \rho \text{ EG} \text{ h d}.
\]

The position, velocity, and acceleration for the center of mass \( C_5 \) are

\[
x_{C_5} = (\text{EG}/2) \cos \phi_5, \quad y_{C_5} = (\text{EG}/2) \sin \phi_5,
\]

\[
\mathbf{r}_{C_5} = x_{C_5} \mathbf{i} + y_{C_5} \mathbf{j}, \quad \mathbf{v}_{C_5} = \dot{x}_{C_5} \mathbf{i} + \dot{y}_{C_5} \mathbf{j}, \quad \text{and} \quad \mathbf{a}_{C_5} = \ddot{x}_{C_5} \mathbf{i} + \ddot{y}_{C_5} \mathbf{j}.
\]
The inertia force is \( \mathbf{F}_{\text{in}5} = -m_5 \mathbf{a}_{C_5} \).

The gravitational force is \( \mathbf{G}_5 = -m_5 \mathbf{g k} \).

The total force on link 5 at \( C_5 \) is \( \mathbf{F}_5 = \mathbf{F}_{\text{in}5} + \mathbf{G}_5 \).

The mass moment of inertia is \( I_{C_5} = m_5 \left( EG^2 + h^2 \right)/12 \).

The moment of inertia is \( M_5 = M_{\text{in}5} = -I_{C_5} \alpha_5 \).

The total force \( \mathbf{F}_5 \) and the moment \( \mathbf{M}_5 \) with Mathematica\textsuperscript{TM} are

\[
\begin{align*}
\text{m5} &= \text{rho \, EG \, h \, d} /. \text{rule}; \\
\text{xC5} &= \text{EG/2 \, Cos[phi5]}; \\
\text{yC5} &= \text{EG/2 \, Sin[phi5]}; \\
\text{rC5} &= \{ \text{xC5, yC5, 0} \}; \\
\text{vC5} &= \text{D[rC5, t]}; \\
\text{aC5} &= \text{D[D[rC5, t], t]}; \\
\text{Fin5} &= -m_5 \, aC5 \ / . \text{rule}; \\
\text{G5} &= \{ 0, -m_5 \mathbf{g}, 0 \} \ / . \text{rule}; \\
\text{F5} &= (\text{Fin5} + \text{G5}) \ / . \text{rule}; \\
\text{IC5} &= m_5 \left( EG^2 + h^2 \right)/12 \ / . \text{rule}; \\
\text{M5} &= -\text{IC5} \, \alpha_5 \ / . \text{rule}; \\
\text{M5e} &= \{ 0, 0, \text{Me} \} \ / . \text{rule}; \\
\text{Print["F5 = ", F5]}; \\
\text{Print["M5 = ", M5]};
\end{align*}
\]

The numerical values are

\[
\begin{align*}
\mathbf{F}_1 &= 0.018 \mathbf{i} - 0.099 \mathbf{j} \text{ N}, \quad \mathbf{M}_1 = 0 \text{ N} \cdot \text{m}, \\
\mathbf{F}_2 &= 0.026 \mathbf{i} - 0.063 \mathbf{j} \text{ N}, \quad \mathbf{M}_2 = -0.00002 \mathbf{k} \text{ N} \cdot \text{m}, \\
\mathbf{F}_3 &= 0.049 \mathbf{i} - 0.333 \mathbf{j} \text{ N}, \quad \mathbf{M}_3 = -0.00621 \mathbf{k} \text{ N} \cdot \text{m}, \\
\mathbf{F}_4 &= -0.036 \mathbf{i} - 0.063 \mathbf{j} \text{ N}, \quad \mathbf{M}_4 = 0.00001 \mathbf{k} \text{ N} \cdot \text{m}, \\
\mathbf{F}_5 &= -0.055 \mathbf{i} - 0.410 \mathbf{j} \text{ N}, \quad \mathbf{M}_5 = 0.00481 \mathbf{k} \text{ N} \cdot \text{m}.
\end{align*}
\]
Joint reaction forces

The diagram representing the mechanism is shown in Fig. 7.11(b) and it has two contours 0-1-2-3-0 and 0-3-4-5-0.

Reaction force $F_{05}$

The rotation joint $E_R$ between the links 0 and 5 is replaced with the unknown reaction force $F_{05}$ (Fig. 7.12)

$$F_{05} = F_{05x} + F_{05y}.$$  

With Mathematica\textsuperscript{TM}, the force $F_{05}$ is written as

$$F_{05Sol} = \{F_{05xSol}, F_{05ySol}, 0\};$$

Following the path $I$, as shown in Fig. 7.12, a force equation is written for the translation joint $D_T$. The projection of all forces, that act on the link 5, onto the sliding direction $r_{DE}$ is zero

$$\sum F^{(5)} \cdot r_{DE} = (F_5 + F_{05}) \cdot r_{DE} = 0,$$  

(7.19)

where $r_{DE} = r_{AE} - r_{AD}$.

Equation (7.19) with Mathematica\textsuperscript{TM} becomes

$$r_{DE} = (r_{E} - r_{D})/.\text{rule};$$
$$eqER1 = (F_5 + F_{05Sol}) \cdot r_{DE} == 0;$$

where the command $\text{a.b}$ gives the scalar product of the vectors $\text{a}$ and $\text{b}$.

Continuing on the path $I$, a moment equation is written for the rotation joint $D_R$

$$\sum M_D^{(4\&5)} = r_{DE} \times F_{05} + r_{DC5} \times F_5 + M_4 + M_5 + M_e = 0,$$  

(7.20)

where $r_{DC5} = r_{AC5} - r_{AD}$.

Equation (7.20) with Mathematica\textsuperscript{TM} gives

$$r_{DC5} = (r_{C5} - r_{D})/.\text{rule};$$
eqER2 = (Cross[rDE, F05Sol] + Cross[rDC5, F5] + M4 + M5 + M5e)[[3]] == 0;

The system of two equations is solved using Mathematica\textsuperscript{TM} command

solF05 = Solve[{eqER1, eqER2}, {F05xSol, F05ySol}];

The following numerical solution is obtained

\[ F_{05} = 268.127 \mathbf{i} + 135.039 \mathbf{j} \text{ N}. \]

Reaction force \( F_{45} \)

The translation joint \( D_T \) between the links 4 and 5 is replaced with the unknown reaction force \( F_{45} \) (Fig. 7.13)

\[ F_{45} = -F_{54} = F_{45x} \mathbf{i} + F_{45y} \mathbf{j}. \]

The position of the application point \( P \) of the force \( F_{45} \) is unknown

\[ r_{AP} = x_P \mathbf{i} + y_P \mathbf{j}, \]

where \( x_P \) and \( y_P \) are the plane coordinates of the point \( P \).

The force \( F_{45} \) and its point of application \( P \) with Mathematica\textsuperscript{TM} is written as

\[ F_{45}\text{Sol} = \{F_{45x}\text{Sol}, \ F_{45y}\text{Sol}, 0 \}; \]
\[ r_{PSol} = \{x_{PSol}, y_{PSol}, 0 \}; \]

Following the path \( I \), Fig. 7.13, a moment equation is written for the rotation joint \( E_R \)

\[ \sum M_E^{(5)} = r_{EP} \times F_{45} + r_{EC_5} \times F_5 + M_5 + M_e = 0, \quad (7.21) \]

where \( r_{EP} = r_{AP} - r_{AE} \), and \( r_{EC_5} = r_{AC_5} - r_{AE} \).

One can write Eq. (7.21) using the Mathematica\textsuperscript{TM} commands

\[ r_{EP} = (r_{PSol} - r_E) /. \text{rule}; \]
\[ r_{EC5} = (r_{C5} - r_E) /. \text{rule}; \]
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\[ eqDT1 = (\text{Cross}[rEP,F45Sol] + \text{Cross}[rEC5,F5] + M5 + M5e)[[3]] == 0; \]

Following the path II, Fig. 7.13, a moment equation is written for the rotation joint \( D_R \)

\[
\sum M_D^{(4)} = r_{DP} \times F_{54} + M_4 = 0, \tag{7.22}
\]

where \( r_{DP} = r_{AP} - r_{AD} \) and \( F_{54} = -F_{45} \).
Equation (7.22) with Mathematica™ is

\[ r_{DP} = (rPSol - rD)/.\text{rule}; \]
\[ eqDT2 = (\text{Cross}[rDP,F54Sol] + M4)[[3]] == 0; \]

The direction of the unknown joint force \( F_{45} \) is perpendicular to the sliding direction \( r_{DE} \)

\[ F_{45} \cdot r_{DE} = 0, \tag{7.23} \]

and using Mathematica™ command

\[ eqDT3 = F45Sol.rDE == 0; \]

The application point \( P \) of the force \( F_{45} \) is located on the direction \( DE \), that is

\[
\frac{y_D - y_E}{x_D - x_E} = \frac{y_P - y_E}{x_P - x_E}. \tag{7.24}
\]

One can write Eq. (7.24) using the Mathematica™ commands

\[ eqDT4 = ((yD-yE)/(xD-xE)/.\text{rule}) == ((yPSol-yE)/(xPSol-xE)/.\text{rule}); \]

The system of four equations is solved using the Mathematica™ command

\[ \text{solF45=Solve[\{eqDT1,eqDT2,eqDT3,eqDT4\}, \{F45xSol,F45ySol,xPSol,yPSol\}];} \]

The following numerical solutions are obtained

\[ F_{45} = -268.072\,i - 134.628\,j \text{ N and } r_{AP} = -0.149\,i + 0.047\,j \text{ m.} \]
Reaction force $F_{34}$

The rotation joint $D_R$ between the links 3 and 4 is replaced with the unknown reaction force $F_{34}$ (Fig. 7.14)

$$F_{34} = -F_{34} = F_{34x} \mathbf{i} + F_{34y} \mathbf{j},$$

and with Mathematica

$$F_{34\text{Sol}} = \{F_{34x\text{Sol}}, F_{34y\text{Sol}}, 0\};$$

Following the path $I$, a force equation can be written for the translation joint $D_T$. The projection of all forces, that act on the link 4, onto the sliding direction $ED$ is zero

$$\sum F^{(4)} \cdot ED = (F_4 + F_{34}) \cdot r_{ED} = 0,$$

(7.25)

where $r_{ED} = r_{AD} - r_{AE}$.

Equation (7.25) using the Mathematica gives

$$r_{ED} = (rD-rE)/.\text{rule};$$

$$\text{eqDR1} = (F_4 + F_{34\text{Sol}}) \cdot r_{ED} == 0;$$

Continuing on the path $I$, Fig. 7.14, a moment equation is written for the rotation joint $E_R$

$$\sum M^{(4\&5)}_E = r_{ED} \times F_{34} + r_{EC4} \times F_4 + r_{EC5} \times F_5 + M_4 + M_5 + M_e = 0.$$ (7.26)

where $r_{EC5} = r_{AC5} - r_{AE}$, and $r_{EC4} = r_{AC4} - r_{AE}$.

Equation (7.26) with Mathematica becomes

$$r_{EC5} = (rC5-rE)/.\text{rule};$$

$$r_{EC4} = (rC4-rE)/.\text{rule};$$

$$\text{eqDR2} = \text{Cross}[r_{EC4}, F_4] + \text{Cross}[r_{EC5}, F_5] + \text{Cross}[r_{ED}, F_{34\text{Sol}}] + M_4 + M_5 + M_e)[[3]] == 0;$$

The system of two equations is solved using the Mathematica commands

$$\text{solF34} = \text{Solve}[\{\text{eqDR1, eqDR2}\}, \{F_{34x\text{Sol}}, F_{34y\text{Sol}}\}];$$
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The following numerical solution is obtained

\[ F_{34} = -268.035 \mathbf{i} - 134.564 \mathbf{j} \text{ N.} \]

Reaction force \( F_{03} \)

The rotation joint \( C_R \) between the links 0 and 3 is replaced with the unknown reaction force \( F_{03} \) (Fig. 7.15)

\[ F_{03} = F_{03x} \mathbf{i} + F_{03y} \mathbf{j}, \]

and with Mathematica™ the force \( F_{03} \) is written as

\[ F_{03Sol} = \{ F_{03xSol}, F_{03ySol}, 0 \}; \]

Following the path I, Fig. 7.15, a force equation is written for the translation joint \( B_T \). The projection of all forces, that act on the link 3, onto the sliding direction \( CD \) is zero

\[ \sum F^{(3)} \cdot r_{CD} = (F_{03} + F_{43} + F_3) \cdot r_{CD} = 0, \]

(7.27)

where \( r_{CD} = r_{AD} - r_{AC} \). Equation (7.27) with Mathematica™ commands is

\[ r_{CD} = (r_D - r_C) /. \text{rule}; \]
\[ \text{eqCR1} = (F_{03Sol} + F_{43} + F_3) \cdot r_{CD} == 0; \]

Continuing on the path II, Fig. 7.15, a moment equation is written for the rotation joint \( B_R \)

\[ \sum M_{B}^{(3&2)} = r_{BC3} \times F_3 + r_{BC} \times F_{03} + r_{BD} \times F_{43} + M_3 + M_2 = 0, \]

(7.28)

where \( r_{BC3} = r_{AC3} - r_{AB}, r_{BC} = r_{AC} - r_{AB}, \) and \( r_{BD} = r_{AD} - r_{AB} \). With Mathematica™ Eq. (7.28) gives

\[ r_{BC3} = (r_{C3} - r_B) /. \text{rule}; \]
\[ r_{BC} = (r_C - r_B) /. \text{rule}; \]
\[ r_{BD} = (r_D - r_B) /. \text{rule}; \]
\[ \text{eqCR2} = (\text{Cross}[r_{BC3}, F_3] + \text{Cross}[r_{BC}, F_{03Sol}] + \text{Cross}[r_{BD}, F_{43}] + M_2 + M_3)[[3]] == 0; \]
To solve the system of two equations the a `Mathematica` command is used

\[ \text{solF03} = \text{Solve}\left[ \lbrace \text{eqCR1}, \text{eqCR2} \rbrace, \lbrace \text{F03Sol}, \text{F03ySol} \rbrace \right]; \]

The following numerical solution is obtained

\[ \text{F}_{03} = -256.71 \mathbf{i} - 272.141 \mathbf{j} \text{ N.} \]

Reaction force \( \text{F}_{23} \)

The translation joint \( B_T \) between the links 2 and 3 is replaced with the unknown reaction force \( \text{F}_{23} \) (Fig. 7.16)

\[ \text{F}_{23} = -\text{F}_{32} = F_{23x} \mathbf{i} + F_{23y} \mathbf{j}. \]

The position of the application point \( Q \) of the force \( \text{F}_{23} \) is unknown

\[ r_{AQ} = x_Q \mathbf{i} + y_Q \mathbf{j}, \]

where \( x_Q \) and \( y_Q \) are the plane coordinates of the point \( Q \).

The force \( \text{F}_{23} \) and its point of application \( Q \) are written in `Mathematica` as

\[
\text{F34Sol} = \{ \text{F34xSol}, \text{F34ySol}, 0 \};
\]
\[
\text{rQSol} = \{ xQSol, yQSol, 0 \};
\]

Following the path \( I \), Fig. 7.16, a moment equation is written for the rotation joint \( C_R \)

\[
\sum M_{C}^{(3)} = r_{CQ} \times \text{F}_{23} + r_{CC_3} \times \text{F}_3 + r_{CD} \times \text{F}_{43} + M_3 = 0, \quad (7.29)
\]

where \( r_{CQ} = r_{AQ} - r_{AC} \), \( r_{CC_3} = r_{AC_3} - r_{AC} \), and \( r_{CD} = r_{AD} - r_{AC} \).

Using `Mathematica`, Eq. (7.29) is written as

\[
\text{rCQ} = (\text{rQSol} - \text{rC}) / . \text{rule};
\]
\[
\text{rCC3} = (\text{rC3} - \text{rC}) / . \text{rule};
\]
\[
\text{rCD} = (\text{rD} - \text{rC}) / . \text{rule};
\]
\[
\text{eqBT1} = (\text{Cross}[\text{rCQ}, \text{F23Sol}] + \text{Cross}[\text{rCC3}, \text{F3}] + \text{Cross}[\text{rCD}, \text{F43}] + \text{M3}) \{[3]\} == 0;
\]
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Following the path II, Fig. 7.16, a moment equation is written for the rotation joint $B_R$

\[ \sum M_B^{(2)} = r_{BQ} \times F_{32} + M_2 = 0, \]  

(7.30)

where $r_{BQ} = r_{AQ} - r_{AB}$. Equation (7.30) with Mathematica\textsuperscript{TM} becomes

```math
rBQ = (rQSol - rB) /. rule;
eqBT2 = (Cross[rBQ, F32Sol] + M2)[[3]] == 0;
```

The direction of the unknown joint force $F_{23}$ is perpendicular to the sliding direction $BC$ and the following relation is written

\[ \mathbf{F}_{23} \cdot \mathbf{r}_{BC} = 0, \]

or with Mathematica\textsuperscript{TM}

```math
eqBT3 = F23Sol.rBC == 0;
```

The application point $Q$ of the force $F_{23}$ is located on the direction $BC$, that is

\[ \frac{y_C - y_B}{x_C - x_B} = \frac{y_C - y_Q}{x_C - x_Q}, \]  

(7.31)

Equation (7.31) with Mathematica\textsuperscript{TM} gives

```math
eqBT4 = ((yC-yB)/(xC-xB) /. rule) == ((yC-yQSol)/(xC-xQSol) /. rule);
```

The system of four equations is solved using the Mathematica\textsuperscript{TM} command

```math
solF23 = Solve[{eqBT1, eqBT2, eqBT3, eqBT4}, 
{F23xSol, F23ySol, xQSol, yQSol}];
```

and the following numerical solutions are obtained

\[ \mathbf{F}_{23} = -11.374\mathbf{i} + 137.91\mathbf{j} \text{ N and } \mathbf{r}_{AQ} = 0.121\mathbf{i} + 0.070\mathbf{j} \text{ m.} \]

Reaction force $\mathbf{F}_{12}$
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The rotation joint $B_R$ between the links 1 and 2 is replaced with the unknown reaction force $F_{12}$ (Fig. 7.17)

$$ F_{12} = -F_{21} = F_{12x} \mathbf{i} + F_{12y} \mathbf{j}, $$

or with Mathematica™

$$ F_{12Sol} = \{F_{12xSol}, F_{12ySol}, 0\}; $$

Following the path $I$, Fig. 7.17, a force equation is written for the translation joint $B_T$. The projection of all forces, that act on the link 2, onto the sliding direction $BC$ is zero

$$ \sum F^{(2)} \cdot r_{BC} = (F_{12} + F_2) \cdot r_{BC} = 0, \quad (7.32) $$

or using Mathematica™

$$ r_{BC} = (r_{C-rB})/.\text{rule}; \quad eqBR1 = (F_{12Sol} + F_2) \cdot r_{BC} = 0; $$

Continuing on the path $I$, a moment equation is written for the rotation joint $C_R$

$$ \sum M^{(2\&3)}_C = r_{CB} \times F_{12} + r_{CC2} \times F_2 + r_{CC3} \times F_3 + r_{CD} \times F_{43} + M_2 + M_3 = 0, \quad (7.33) $$

where $r_{CB} = r_{AB} - r_{AC}$, $r_{CC2} = r_{AC2} - r_{AC}$, $r_{CC3} = r_{AC3} - r_{AC}$, and $r_{CD} = r_{AD} - r_{AC}$.

Using the Mathematica™ commands Eq. (7.33) gives

$$ r_{CB} = (r_{B-rC})/.\text{rule}; \quad r_{CC2} = (r_{C2-rC})/.\text{rule}; \quad r_{CC3} = (r_{C3-rC})/.\text{rule}; \quad r_{CD} = (r_{D-rC})/.\text{rule}; \quad eqBR2 = (\text{Cross}[r_{CB}, F_{12Sol}] + \text{Cross}[r_{CC2}, F_2] + \text{Cross}[r_{CC3}, F_3] + \text{Cross}[r_{CD}, F_{43}] + M_2 + M_3)[[3]] = 0; $$

The system of two equations is solved using the Mathematica™ command
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solF12 = Solve[{eqBR1, eqBR2}, {F12xSol, F12ySol}];

and the following numerical solution is obtained

\[ F_{12} = -11.4011 + 137.974 j \text{ N.} \]

The motor moment \( M_m \)

The motor moment needed for the dynamic equilibrium of the mechanism is \( M_m = M_m k \) (Fig. 7.18) and with Mathematica

\[ M_{1mSol} = \{0, 0, MmSol\}; \]

Following the path \( I \), Fig. 7.18, a moment equation is written for the rotation joint \( A_R \)

\[ \sum M_A^{(i)} = r_{AB} \times F_{21} + r_{AC1} \times F_1 + M_1 + M_m = 0. \]  (7.34)

Equation (7.34) is solved using Mathematica

\[ eqMA = (\text{Cross}[rAB, F21] + \text{Cross}[rAC1, F1] + M1 + M_{1mSol})[[3]] == 0; \]
\[ solMm = \text{Solve}[eqMA, MmSol]; \]
\[ Mm = MmSol/.solMm[[1]]; \]
\[ M1m = \{0, 0, Mm\}; \]
\[ \text{Print}["Mm = ", Mm]; \]

The numerical solution is

\[ M_m = 17.533 k \text{ N.m.} \]

Reaction force \( F_{01} \)

The rotation joint \( A_R \) between the links 0 and 1 is replaced with the unknown reaction force \( F_{01} \) (Fig. 7.19)

\[ F_{01} = -F_{10} = F_{01x1} + F_{01yJ}, \]

and with Mathematica
I.7 Simulation of Kinematic Chains with Mathematica™

Following the path $I$, Fig. 7.19, a moment equation is written for the rotation joint $B_R$

$$
\sum M_B^{(1)} = r_{BA} \times \mathbf{F}_{01} + r_{BC1} \times \mathbf{F}_1 + M_1 + M_m = 0,
$$

(7.35)

where $r_{BA} = -r_{AB}$, and $r_{BC1} = r_{AC1} - r_{AB}$.

Equation (7.35) using the Mathematica™ gives

\begin{verbatim}
rBA=-rB/.rule;
rBC1=(rC1-rB)/.rule;
eqAR1=(Cross[rBA,F01Sol]+Cross[rBC1,F1]+M1+M1m)[[3]]==0;
\end{verbatim}

Continuing on the path $I$, Fig. 7.19, a force equation is written for the translation joint $B_T$. The projection of all forces, that act on the links 1 and 2, onto the sliding direction $BC$ is zero

$$
\sum \mathbf{F}^{(1\&2)} \cdot r_{BC} = (\mathbf{F}_{01} + \mathbf{F}_1 + \mathbf{F}_2) \cdot r_{BC} = 0,
$$

(7.36)

or with Mathematica™

\begin{verbatim}
eqAR2=(F01Sol+F1+F2).rBC==0;
\end{verbatim}

The system of two equations is solved using the Mathematica™ command

\begin{verbatim}
solF01=Solve[{eqAR1,eqAR2},{F01xSol,F01ySol}];
\end{verbatim}

The following numerical solution is obtained

$$
\mathbf{F}_{01} = -11.419 \mathbf{i} + 138.073 \mathbf{j} \text{ N}.
$$

The Mathematica™ program for the dynamic force analysis is presented in Program 7.6.
7.5 Problems

7.1 Referring to Example 3.1 (Fig. 3.11). Write a Mathematica\textsuperscript{TM} program for the position analysis of the mechanism.

7.2 Referring to Example 3.2 (Fig. 3.12). Write a Mathematica\textsuperscript{TM} program for the position analysis of the mechanism.

7.3 Referring to Example 3.3 (Fig. 3.15). Write a Mathematica\textsuperscript{TM} program for the position analysis of the mechanism.

7.4 Referring to Problem 3.4 (Fig. 3.19). Write a Mathematica\textsuperscript{TM} program for the position analysis of the mechanism.

7.5 Referring to Problem 3.5 (Fig. 3.20). Write a Mathematica\textsuperscript{TM} program for the position analysis of the mechanism.

7.5 Referring to Problem 3.11 (Fig. 3.26). Write a Mathematica\textsuperscript{TM} program for the position analysis of the mechanism.

7.7 Referring to Example 4.1 (Fig. 4.7). Write a Mathematica\textsuperscript{TM} program for the velocity and acceleration analysis of the mechanism.

7.8 Referring to Example 4.2 (Fig. 4.8). Write a Mathematica\textsuperscript{TM} program for the velocity and acceleration analysis of the mechanism.

7.9 Referring to Problem 4.1 (Fig. 3.16(a)). Write a Mathematica\textsuperscript{TM} program for the velocity and acceleration analysis of the mechanism.

7.10 Referring to Problem 5.1 (Fig. 3.16(a)). Write a Mathematica\textsuperscript{TM} program for the velocity and acceleration analysis of the mechanism using the contour equations method.

7.11 Referring to Problem 4.3 (Fig. 4.10). Write a Mathematica\textsuperscript{TM} program for the velocity and acceleration analysis of the mechanism.

7.12 Referring to Problem 5.3 (Fig. 3.10). Write a Mathematica\textsuperscript{TM} program for the velocity and acceleration analysis of the mechanism using the contour equations method.

7.13 Referring to Problem 4.4 (Fig. 3.19). Write a Mathematica\textsuperscript{TM} program for the velocity and acceleration analysis of the mechanism.
7.14 Referring to Problem 5.4 (Fig. 3.19). Write a Mathematica™ program for the velocity and acceleration analysis of the mechanism using the contour equations method.

7.15 Referring to Problem 6.3 (Fig. 4.10). Write a Mathematica™ program for the equilibrium moment and the joint forces of the mechanism.

7.16 Referring to Problem 6.16 (Fig. 3.31). Write a Mathematica™ program for the equilibrium moment and the joint forces of the mechanism.

7.17 Referring to Problem 6.18 (Fig. 3.33). Write a Mathematica™ program for the equilibrium moment and the joint forces of the mechanism.

7.18 Referring to Problem 6.24 (Fig. 3.11). Write a Mathematica™ program for the equilibrium moment and the joint forces of the mechanism.

7.19 Referring to Problem 6.25 (Fig. 3.12). Write a Mathematica™ program for the equilibrium moment and the joint forces of the mechanism.

7.20 Referring to Problem 6.26 (Fig. 3.15). Write a Mathematica™ program for the equilibrium moment and the joint forces of the mechanism.
References


[60] * * * , *The theory of mechanisms and machines (Teoria mehanizmov i masin)*, Vassaia scola, Minsc, 1970.

Figure captions

Figure 7.1 R-RTR-RTR mechanism
Figure 7.2 Solutions for the position of the joint $D$ for $0 \leq \phi \leq 90^\circ$ ($x_D \leq x_C$)
Figure 7.3 Solutions for the position of the joint $D$ for $90^\circ < \phi \leq 180^\circ$ ($x_D \geq x_C$)
Figure 7.4 Solutions for the position of the joint $D$ for $180^\circ < \phi < 270^\circ$ ($x_D \geq x_C$)
Figure 7.5 Solutions for the position of the joint $D$ for $270^\circ \leq \phi \leq 360^\circ$ ($x_D \leq x_C$)
Figure 7.6 Graph of the mechanism for a complete rotation, $0 \leq \phi \leq 360^\circ$
Figure 7.7 Distance condition for position analysis: $d^I_k < d^{II}_k \Rightarrow D_{k+1} = D^I_{k+1}$
Figure 7.8 (a) R-RTR-RTR mechanism (b) contour diagram
Figure 7.9 First independent contour
Figure 7.10 Second independent contour
Figure 7.11 Forces and moments for R-RTR-RTR mechanism
Figure 7.12 Rotation joint $E_R$ and reaction force $F_{05}$
Figure 7.13 Translation joint $D_T$ and reaction force $F_{45}$
Figure 7.14 Rotation joint $D_R$ and reaction force $F_{34}$
Figure 7.15 Rotation joint $C_R$ and reaction force $F_{03}$
Figure 7.16 Translation joint $B_T$ and reaction force $F_{23}$
Figure 7.17 Rotation joint $B_R$ and reaction force $F_{12}$
Figure 7.18 Dynamic equilibrium moment $M_m$
Figure 7.19 Rotation joint $A_R$ and reaction force $F_{01}$